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H₂ week in Toulouse, February 26th-29th 2024

Running high-pressure experiments with hydrogen: *challenges, flame stabilization, diagnostics opportunities*

Thibault F. Guiberti



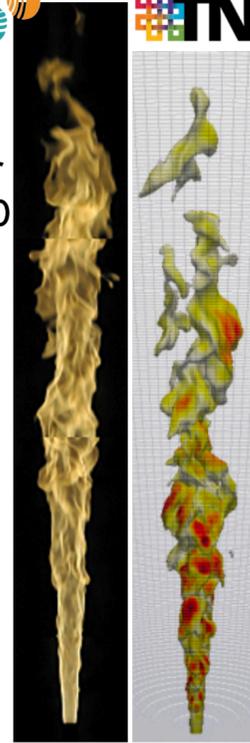
High pressure turbulent flames at KAUST



Non-premixed flames



NH₃-H₂-N₂
up to 5 bar
Re = 11,200

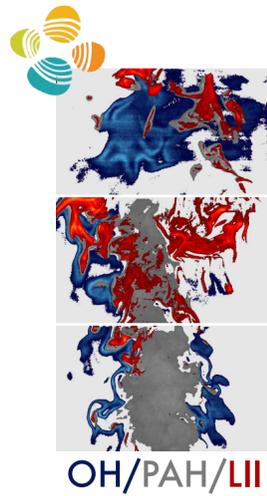


LES

C₂H₄-N₂
up to 7 bar and
Re = 50,000

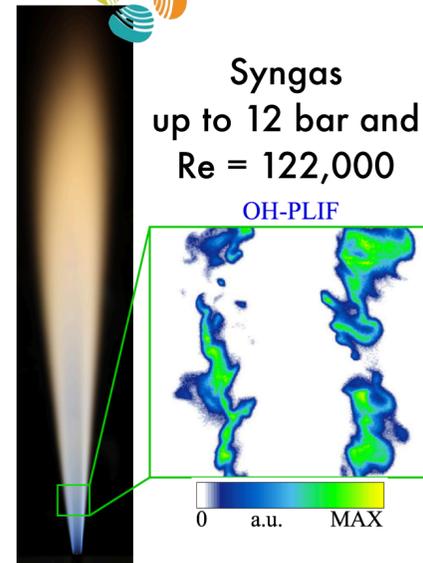


Soot



OH/PAH/LII

Syngas
up to 12 bar and
Re = 122,000

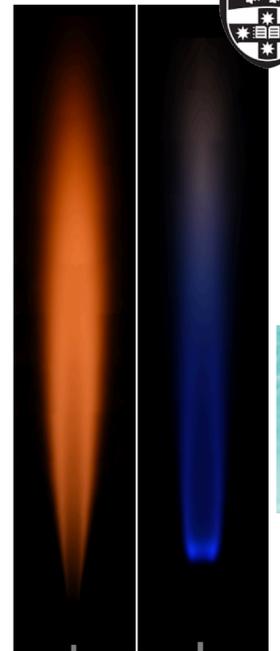


Structure

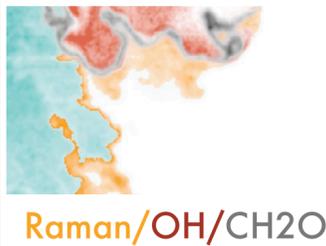
OH-PLIF

0 a.u. MAX

CH₄ & C₂H₆
up to 10 bar and
Re = 50,000



Stabilization



Raman/OH/CH₂O

Premixed flames

NH₃-H₂
up to 5 bar

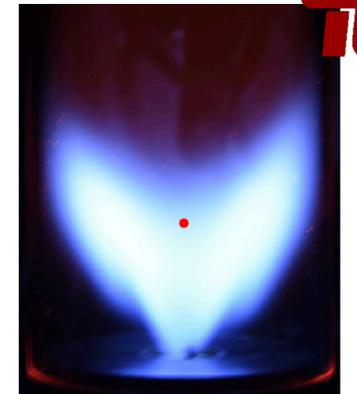


Ammonia blends

NH₃-CH₄
up to 5 bar

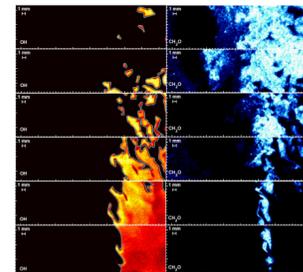


CH₄ & C₃H₈
up to 5 bar and
P = 20 kW



Flame dynamics

CH₄
up to 3 bar and
Re = 170,000



OH CH₂O



Blow-off

H₂
up to 10 bar



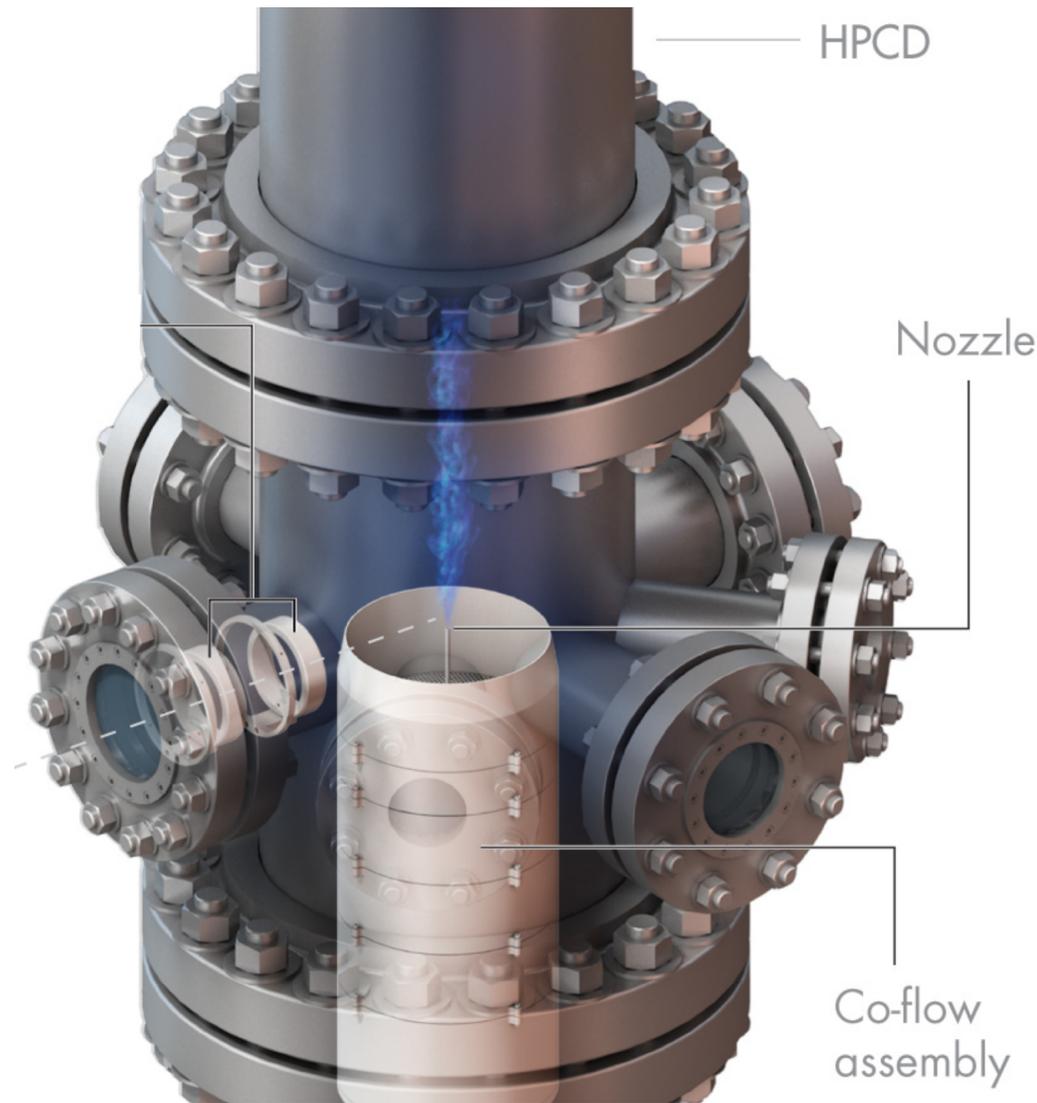
Stabilization



High pressure turbulent H_2 flames at KAUST



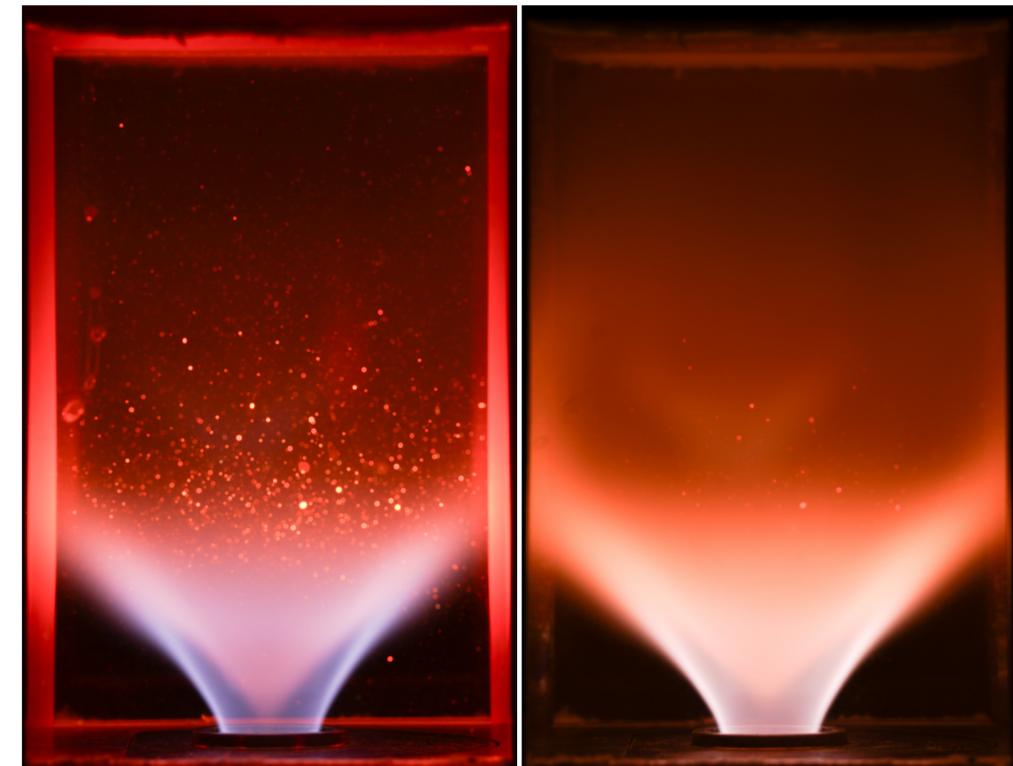
H_2 - N_2 non-premixed jet flames
up to 12 bar
up to $Re = 83,000$



H_2 -air dual swirl flames

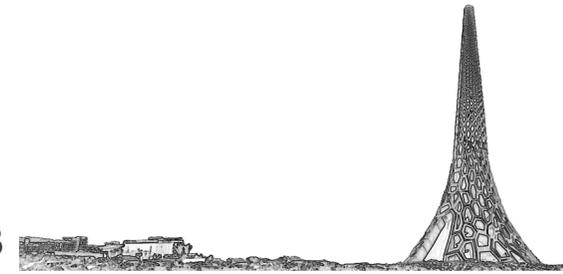
4 bar

10 bar



HYLON-KAUST burner

These flames have in common the non-premixed nature of their injection system

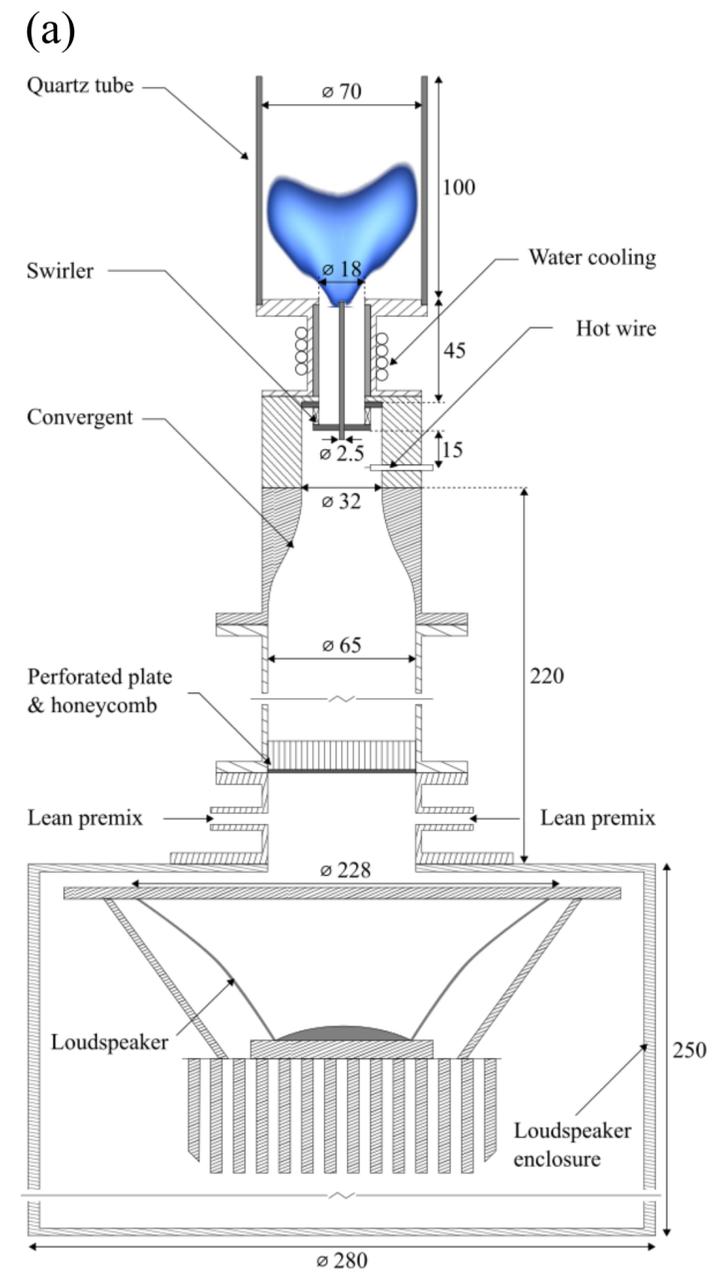


The dreaded flashback...



During FTF measurements in premixed CH₄/C₃H₈-air flames at 5 bar:

Failure of the plenum screws



Deformation of the aluminum rim



At pressure the severity of the damages due to flashback increase!

Di Sabatino et al. Combust. Flame 2018(193)





The dreaded flashback...

At pressure the severity of the damages due to flashback increase!

Deflagration @ 1 bar for stoichiometric CH₄-air: $\Delta P \approx 1 \text{ bar} \times (2225\text{K}/300\text{K} - 1) = 6.4 \text{ bar}$

Deflagration @ 5 bar for stoichiometric CH₄-air: $\Delta P \approx 5 \text{ bar} \times (2257\text{K}/300\text{K} - 1) = 32.6 \text{ bar}$

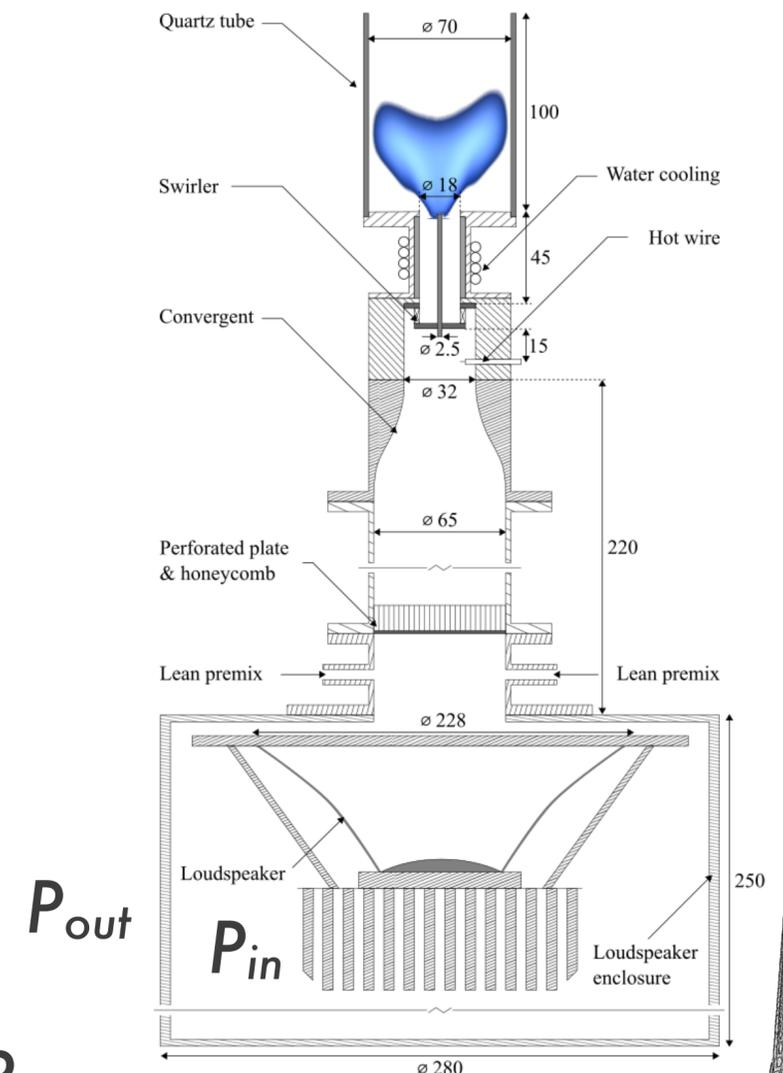
Deflagration @ 1 bar for stoichiometric H₂-air: $\Delta P \approx 6.9 \text{ bar}$

Deflagration @ 5 bar for stoichiometric H₂-air: $\Delta P \approx 35.4 \text{ bar}$

This assumes an adiabatic flashback process

This also assumes that expanded gases do not escape during flashback

- ▶ H₂ flashbacks are more severe due to the larger flame speed



$$\Delta P = P_{in} - P_{out}$$

The dreaded flashback...



Consequences of flashback may be even more dramatic if transition to detonation occurs

Detonation @ 1 bar for stoichiometric CH₄-air: $\Delta P \approx 17.1 - 1 \text{ bar} = 16.1 \text{ bar}$

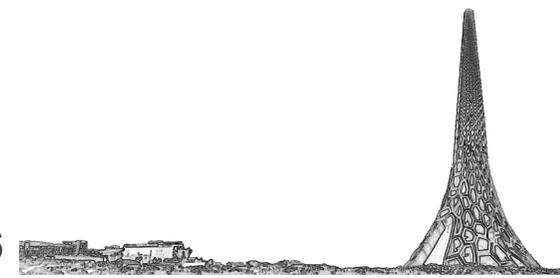
Detonation @ 5 bar for stoichiometric CH₄-air: $\Delta P \approx 87.1 - 5 \text{ bar} = 82.1 \text{ bar}$

**Very unlikely
to occur**

Detonation @ 1 bar for stoichiometric H₂-air: $\Delta P \approx 15.5 - 1 \text{ bar} = 14.5 \text{ bar}$

Detonation @ 5 bar for stoichiometric H₂-air: $\Delta P \approx 79.3 - 5 \text{ bar} = 74.3 \text{ bar}$

Detonation @ 10 bar for stoichiometric H₂-air: $\Delta P \approx 160.0 - 10 \text{ bar} = 150.0 \text{ bar}$





The dreaded flashback...

For a detonation to occur, suitable conditions must be met

This includes providing enough space for detonation cells to develop

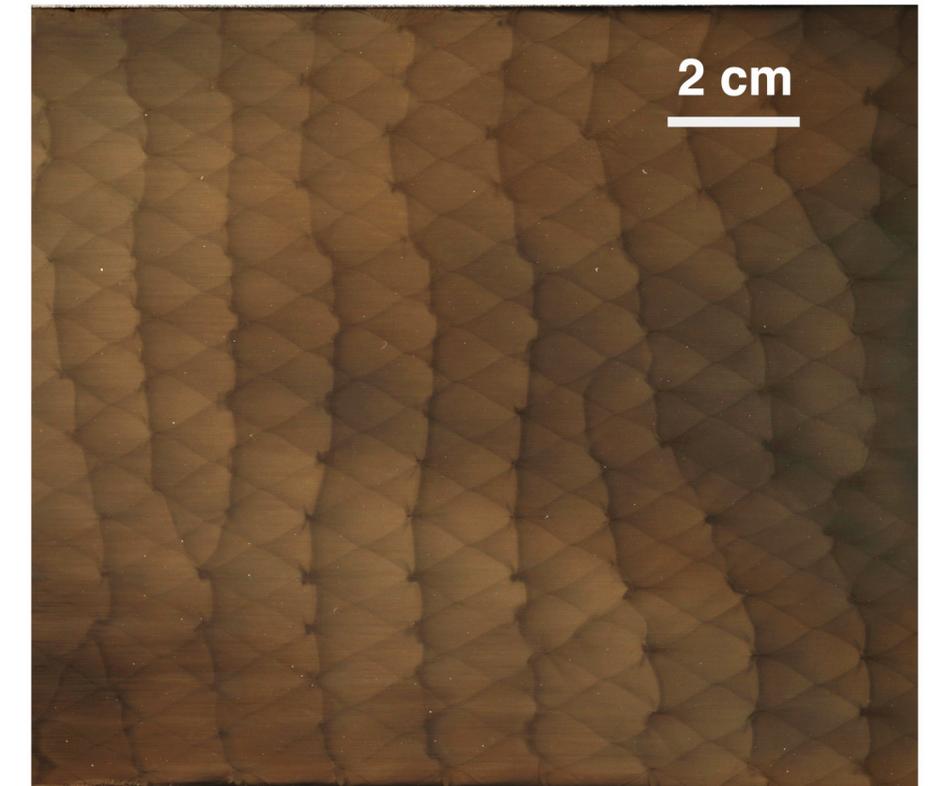
approx. cell width @ 293 K	1 bar	5 bar	10 bar
CH₄-air $\phi = 1$	300 mm	N.A.	N.A.
H₂-air $\phi = 1$	10 mm	5 mm	3 mm
H₂-air $\phi = 0.65$	40 mm	9 mm	8 mm

Kaneshige & Sheperd Detonation database 1999

Curtis A. Babbie Thesis 2015

Soot foil

2H₂-O₂-3.76Ar @ 20 kPa & 293 K



Alicherif et al. Combust. Flame (under review)

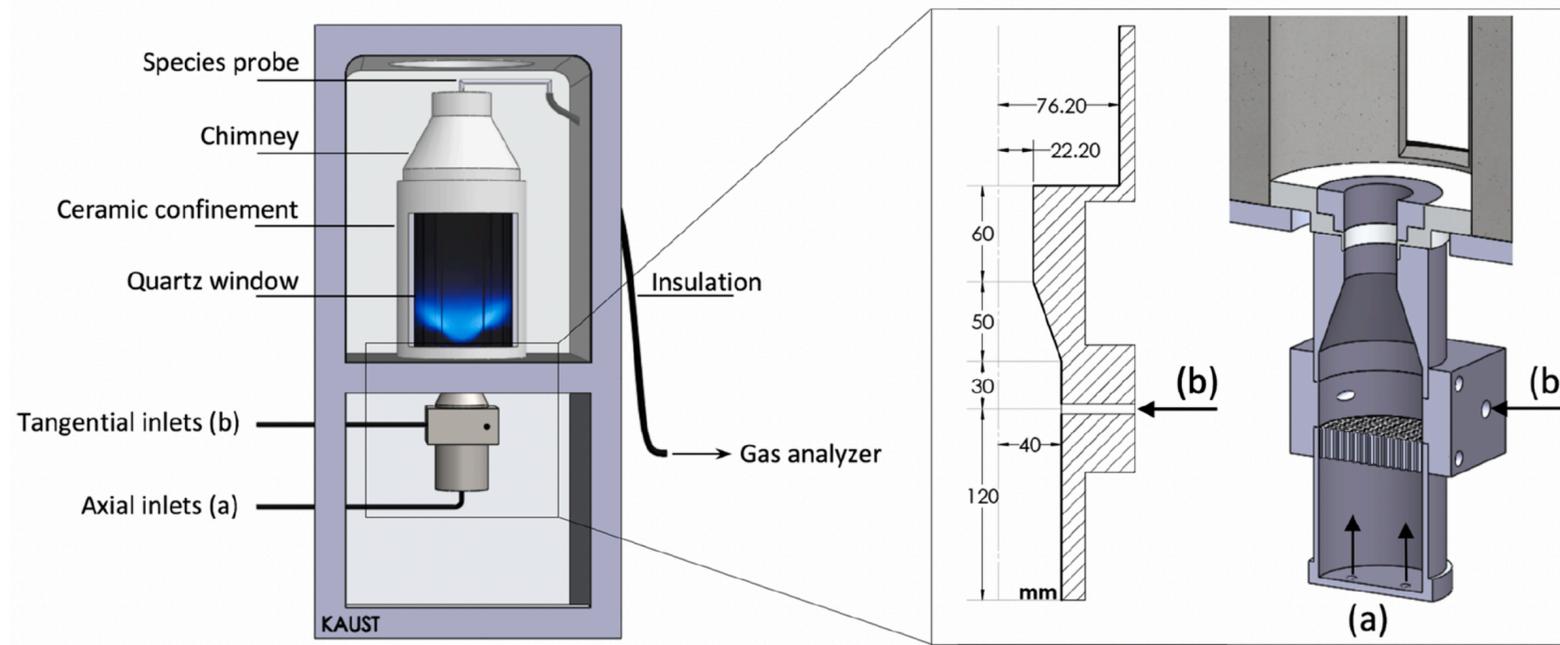
In case of flashback, detonation may occur within the burner for premixed H₂-air mixtures...



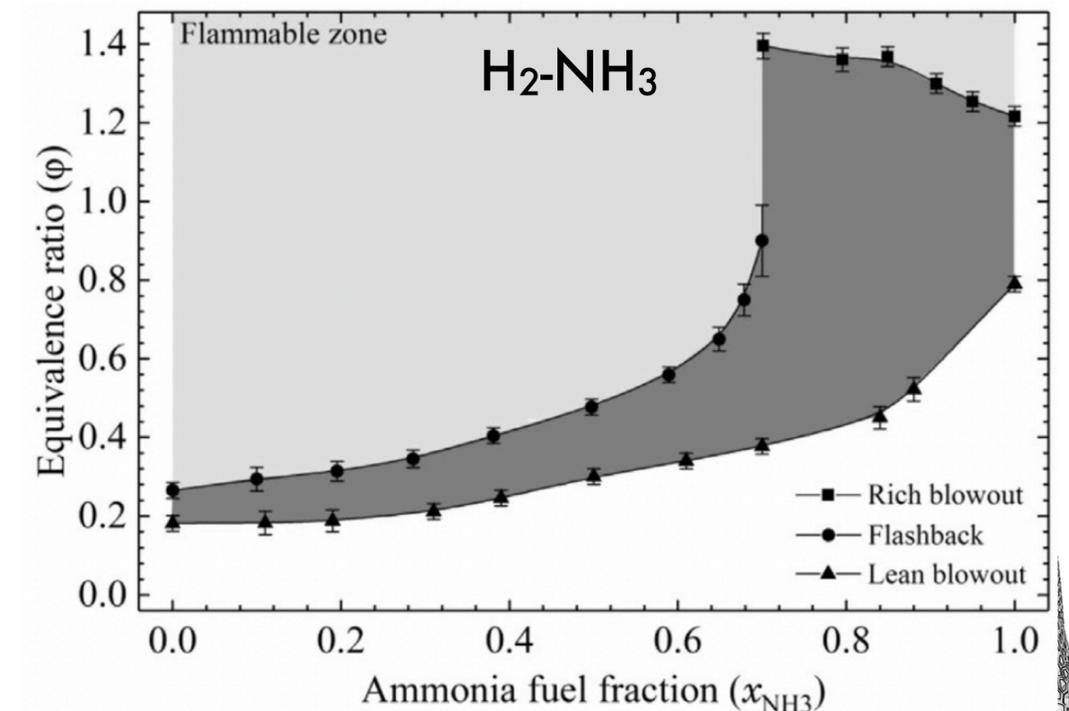
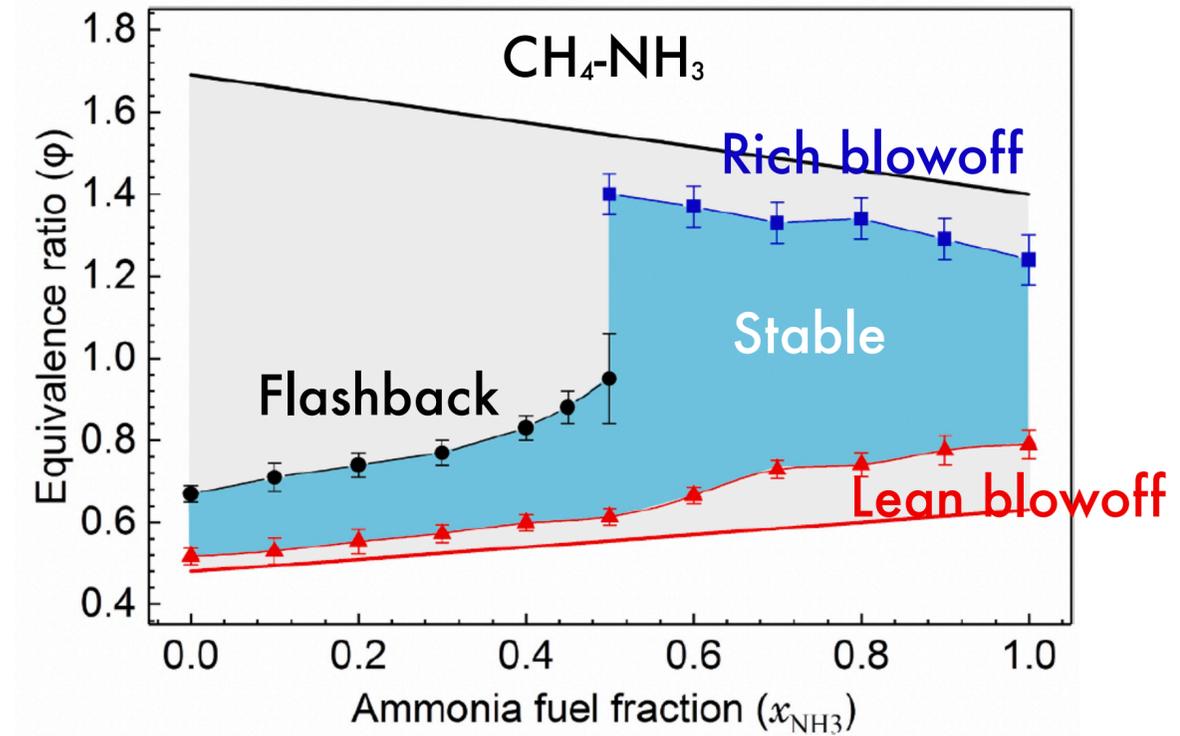


The dreaded flashback...

Because of their comparatively high turbulent flame speed and small quenching distance, H₂ flames are prone to flashback



Lab-scale swirl burner designed to withstand flashback

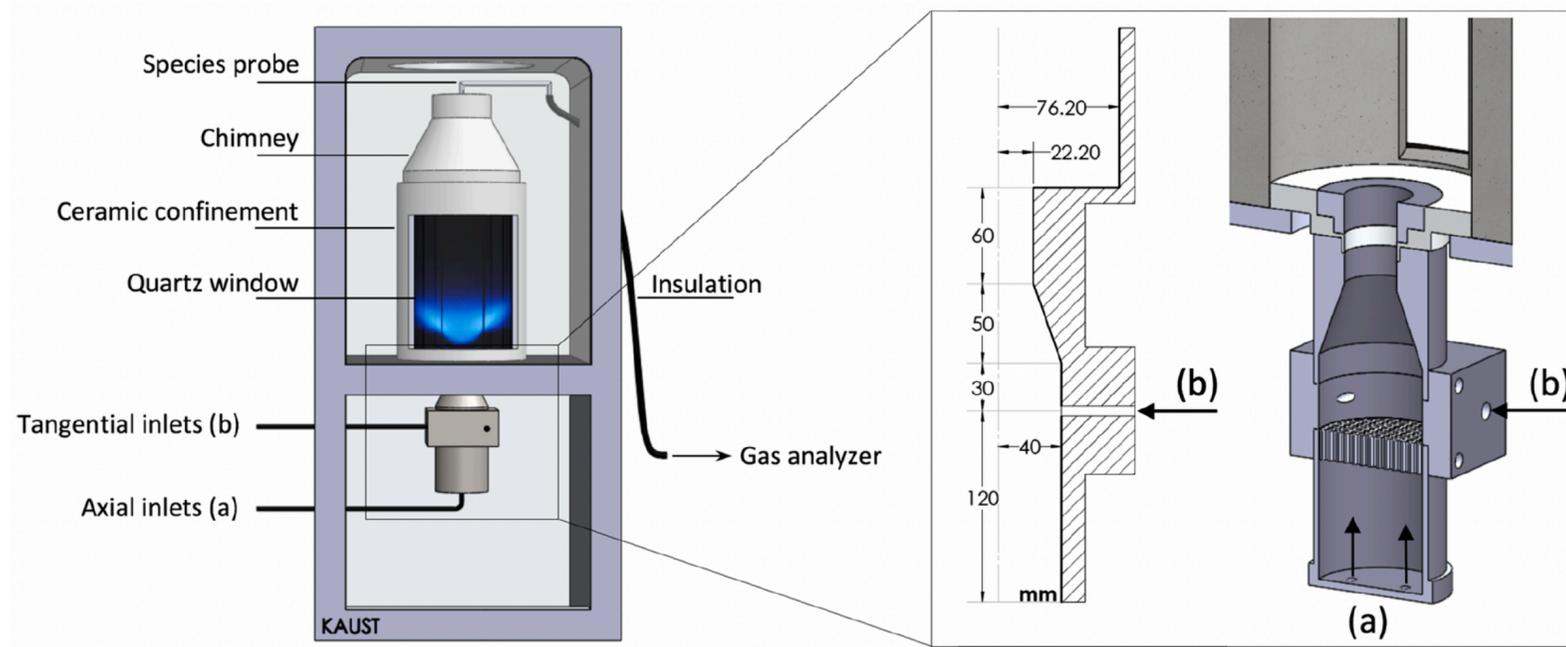


Khateeb et al. *Exp. Therm. Fluid Sci.* 2020(114)

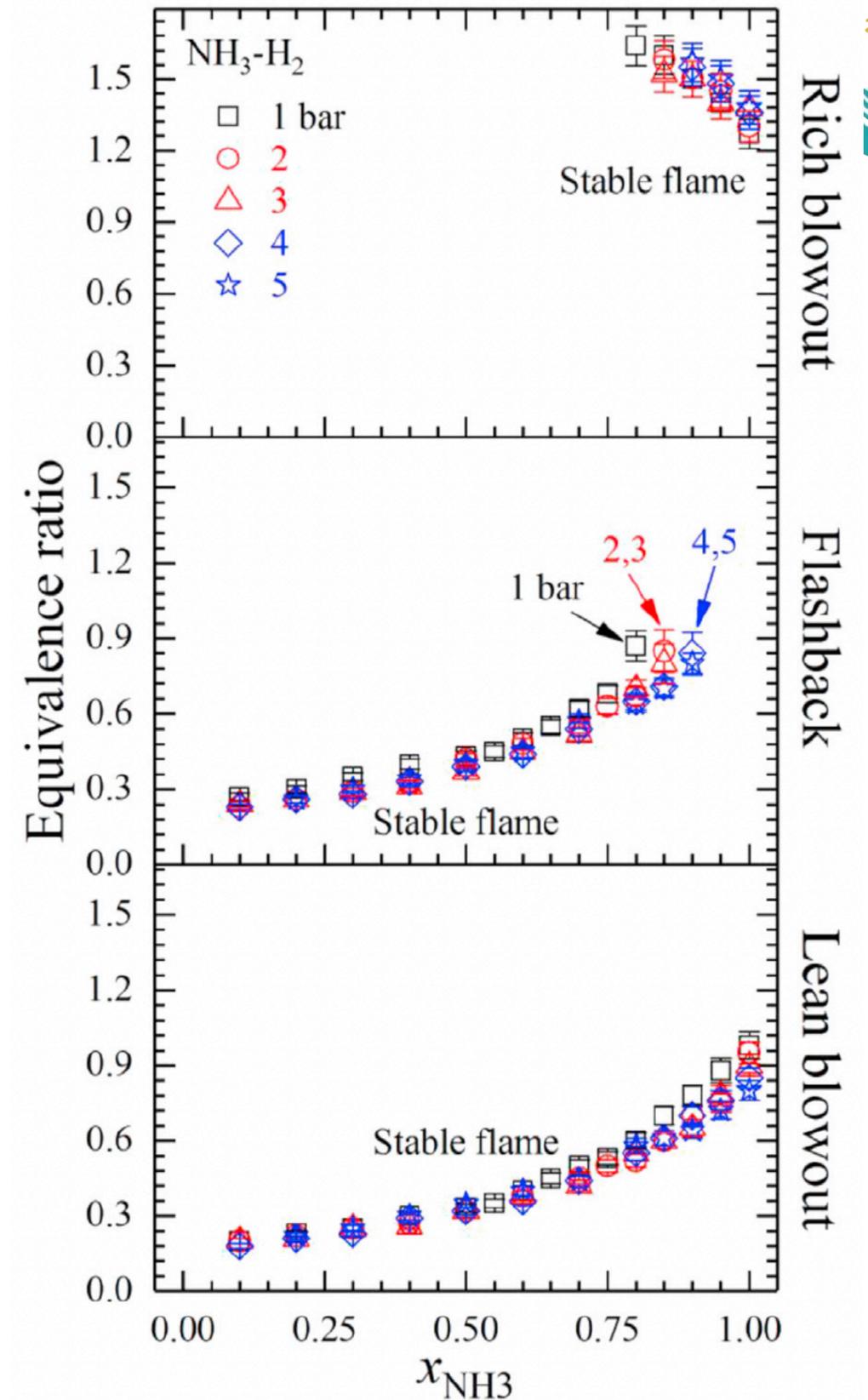
Khateeb et al. *Int. J. Hydrogen Energy.* 2020(45)

The dreaded flashback...

Increasing pressure may promote flashback through effects of increasing turbulence intensity



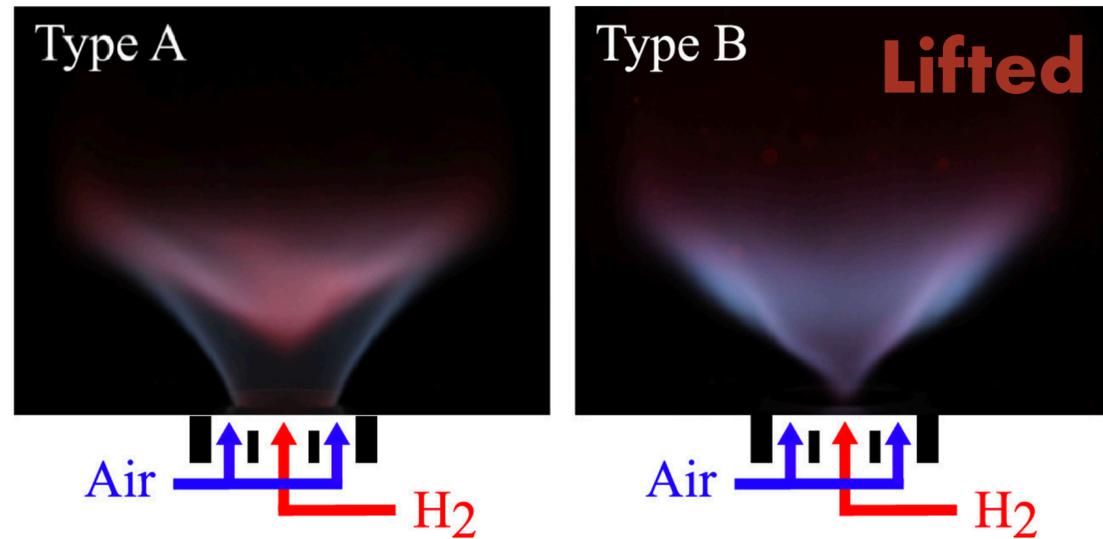
Lab-scale swirl burner designed to withstand flashback



The convenience of dual swirl H₂-air flames



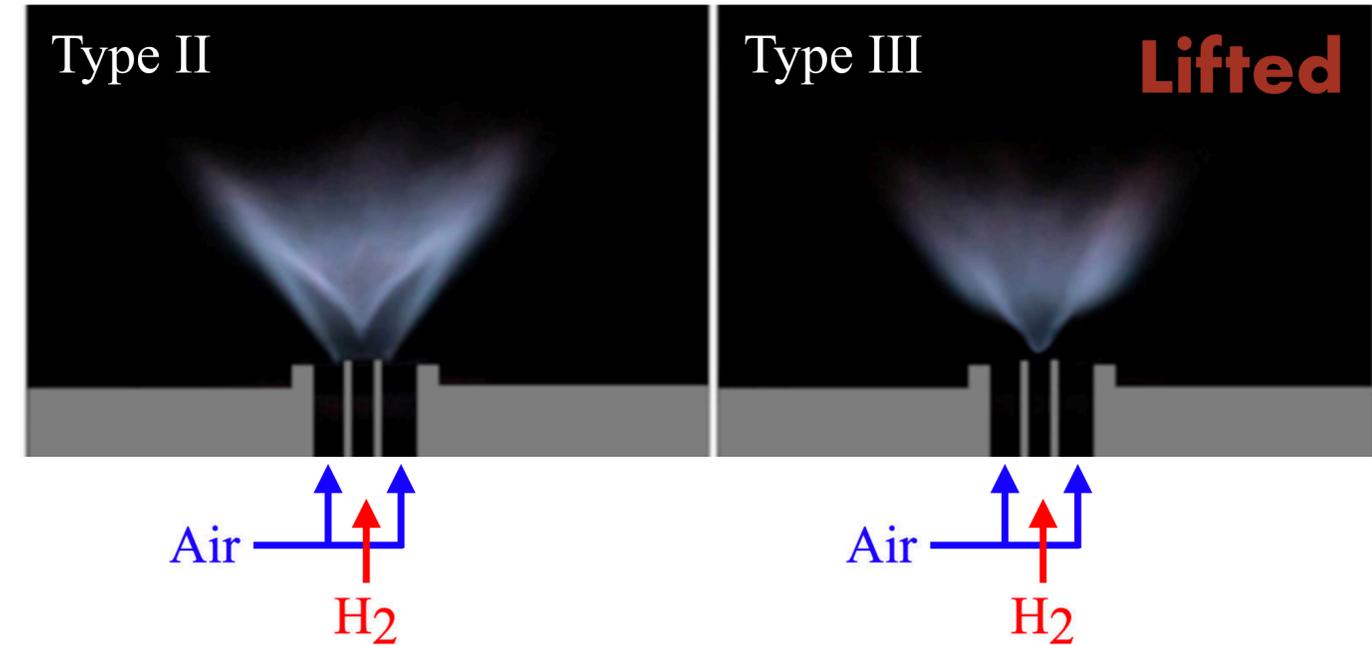
H₂-air flames stabilized with a dual swirl co-axial burner are promising candidates for the decarbonization of gas turbines



Marragou et al. Combust. Flame 2023

Marragou et al. Int. Proc. Combust. Inst. 2023

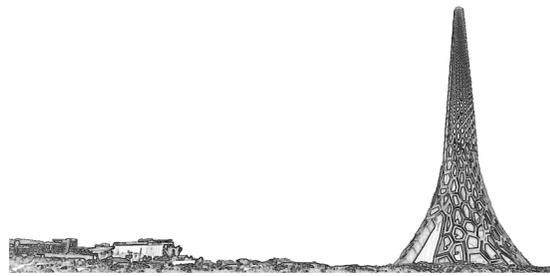
Marragou et al. Int. J. Hydrogen Energ. 2022



Leroy et al. J. Eng. Gas Turbines Power 2023

This technology is capable of achieving low NO_x emissions while providing flashback protection and minimizing injector wall temperature

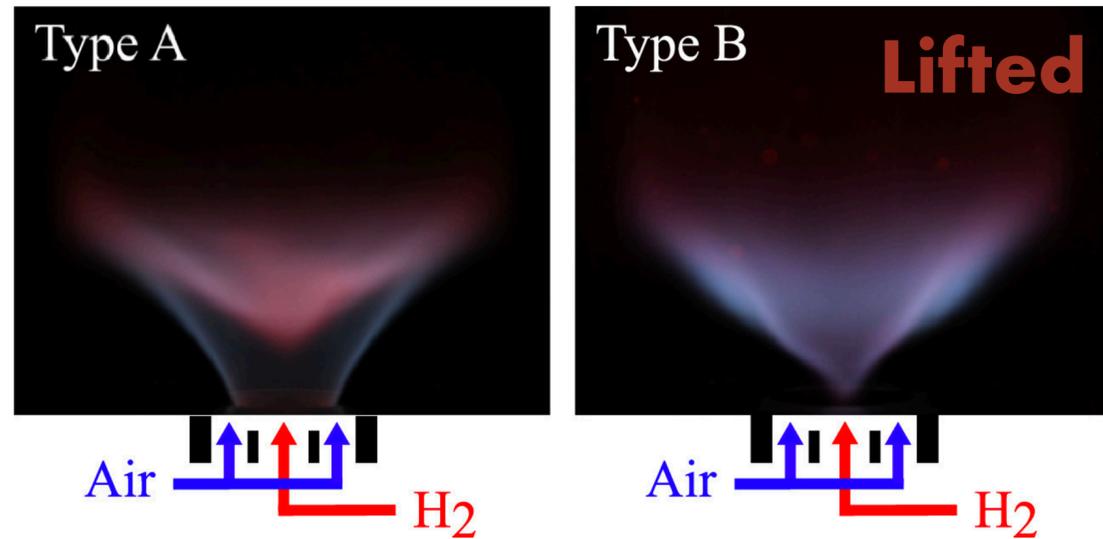
However, this implies maintaining the flame in the lifted regime



The convenience of dual swirl H₂-air flames



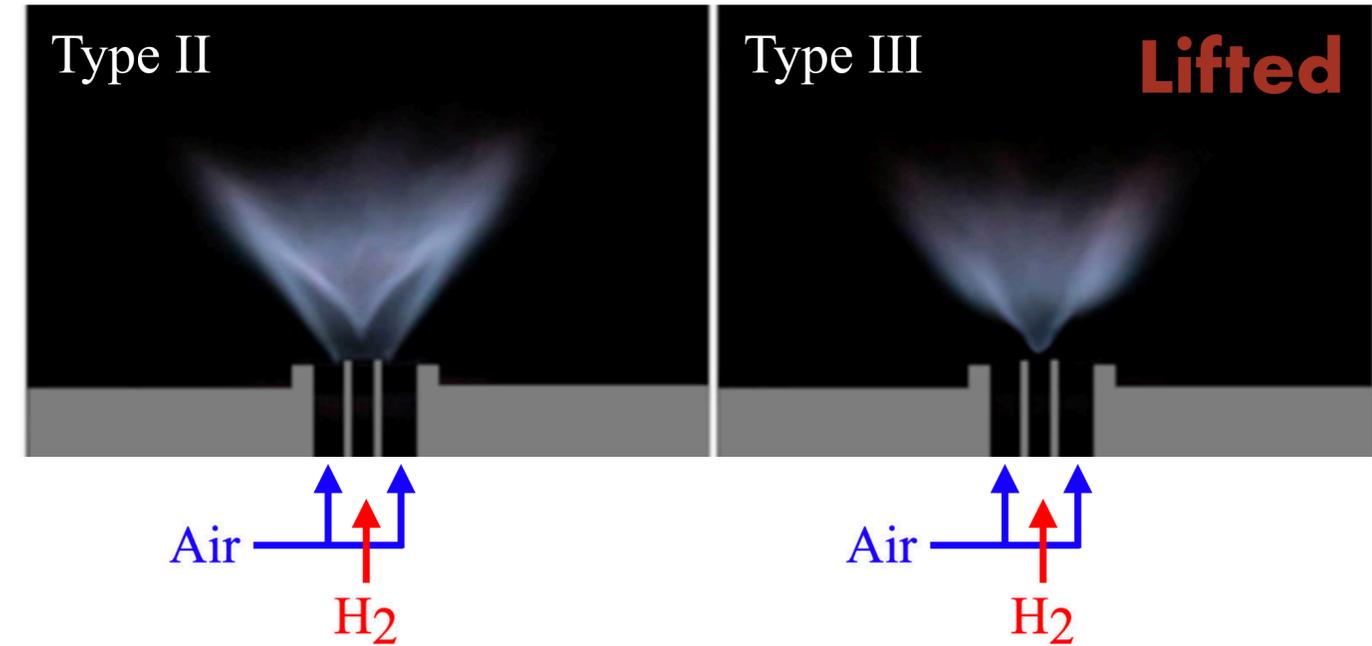
H₂-air flames stabilized with a dual swirl co-axial burner are promising candidates for the decarbonization of gas turbines



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Marragou et al. Int. Proc. Combust. Inst. 2023

Marragou et al. Int. J. Hydrogen Energ. 2022



Leroy et al. J. Eng. Gas Turbines Power 2023

There is a need to understand the physical mechanism(s) leading to the detachment and reattachment of these flames

With practical applications in mind, effects of elevating pressure must be understood too

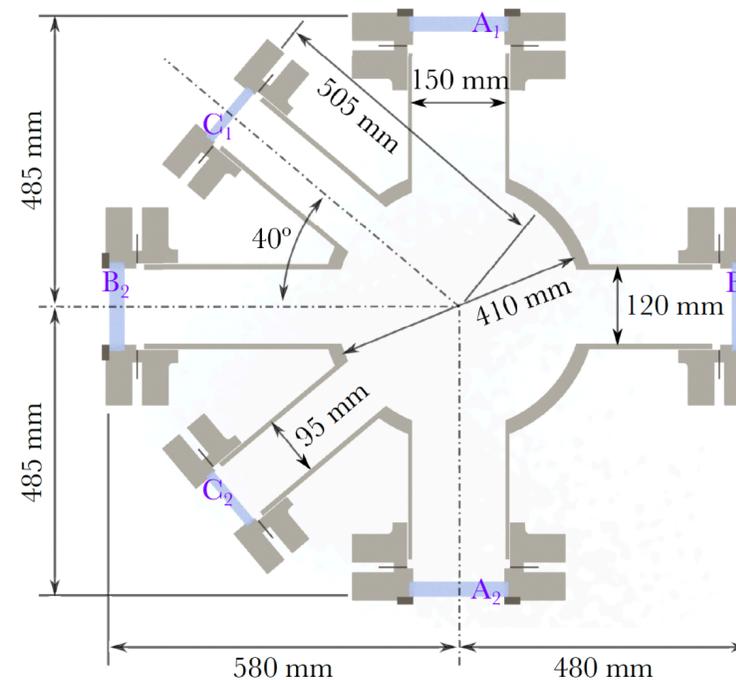


KAUST's High Pressure Combustion Duct (HPCD)

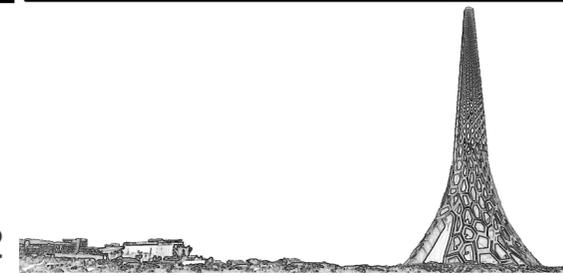
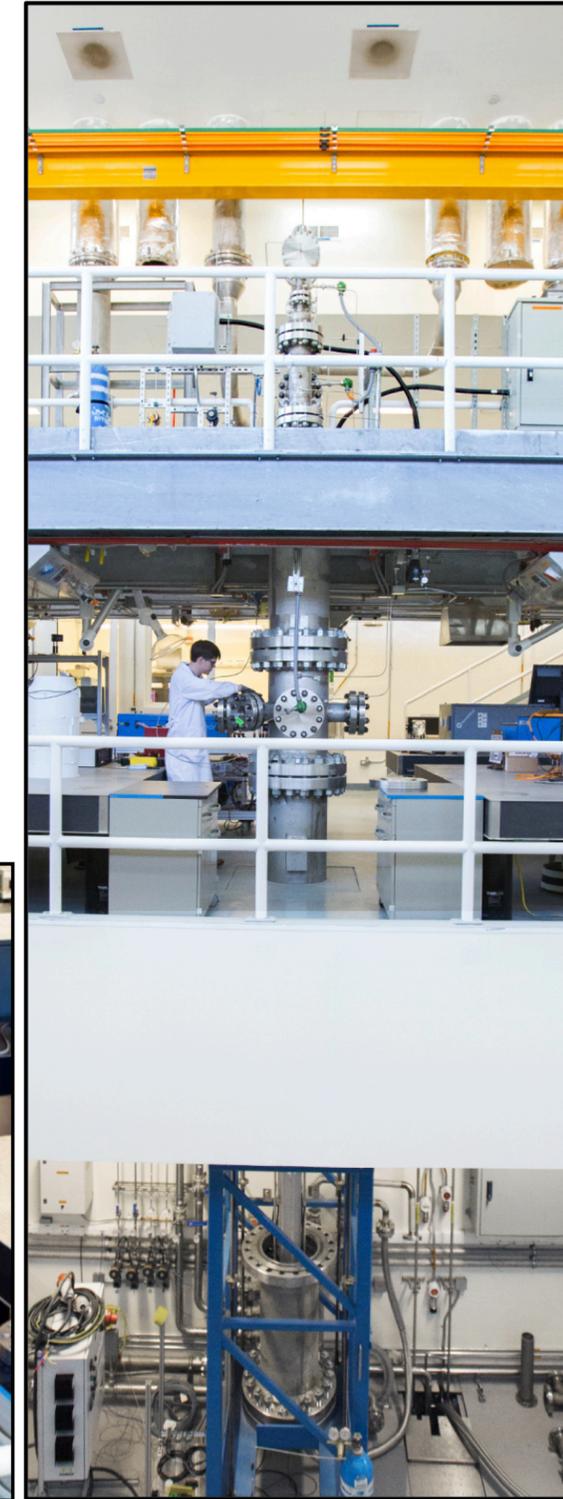
The HPCD is one of the high-pressure test-rigs available at CCRC in KAUST

9-m tall & vertical

*40 bar
>100 kW @ steady state*



Ample optical access





Turbulent non-premixed jet flames

Turbulent non-premixed jet flames can be found in three states:

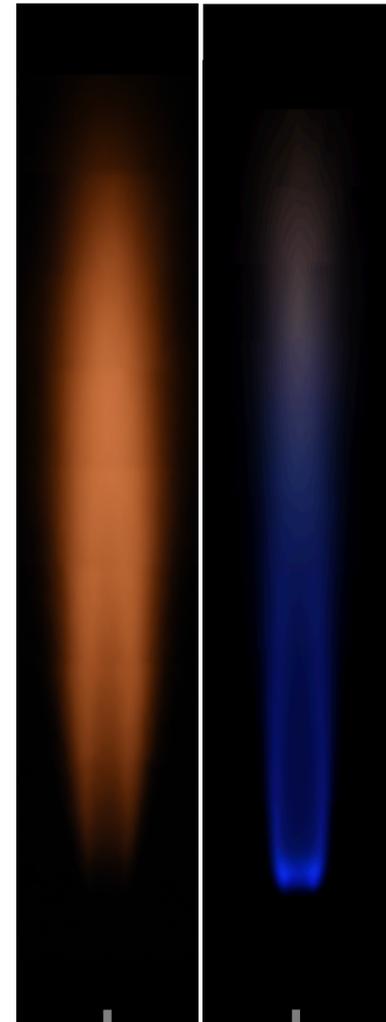
Attached



Detachment



Lifted



blow-off



Blown-off



Can we predict these transitions for any fuel, pressure, nozzle geometry?

This is important for flares, punctured fuel tanks, furnaces, etc... **and H₂ swirl flames**



Flame detachment



From previous work at 1 bar (e.g. Takahashi and Schroll, Proc. Combust. Inst., 1991), two detachment mechanisms are typically observed:

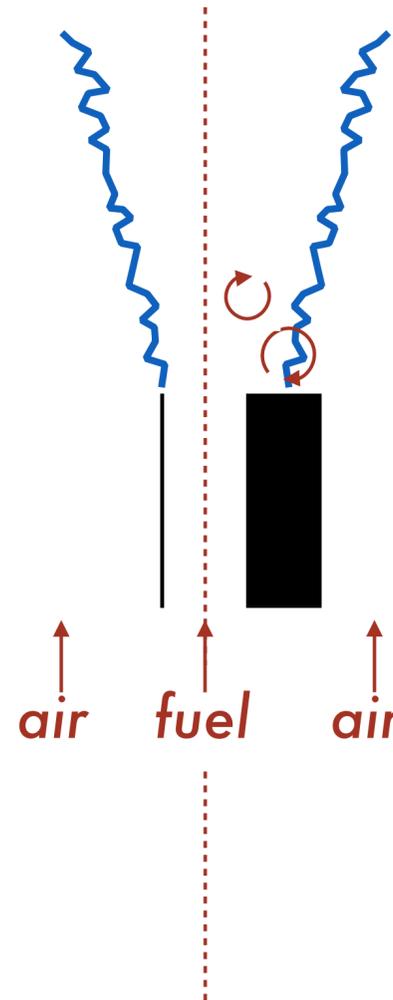
δ decreases with P!

$$\delta \gg t$$

If the flame thickness δ is larger than the rim thickness t :

The flame edge is exposed to the incoming flow

Aerodynamic detachment



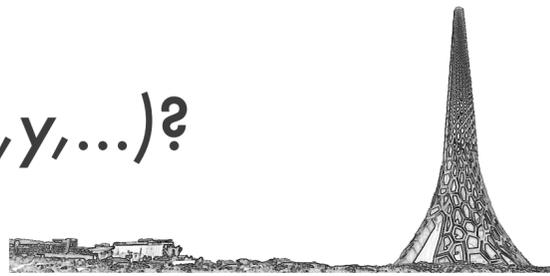
$$\delta \ll t$$

If the flame thickness δ is smaller than the rim thickness t :

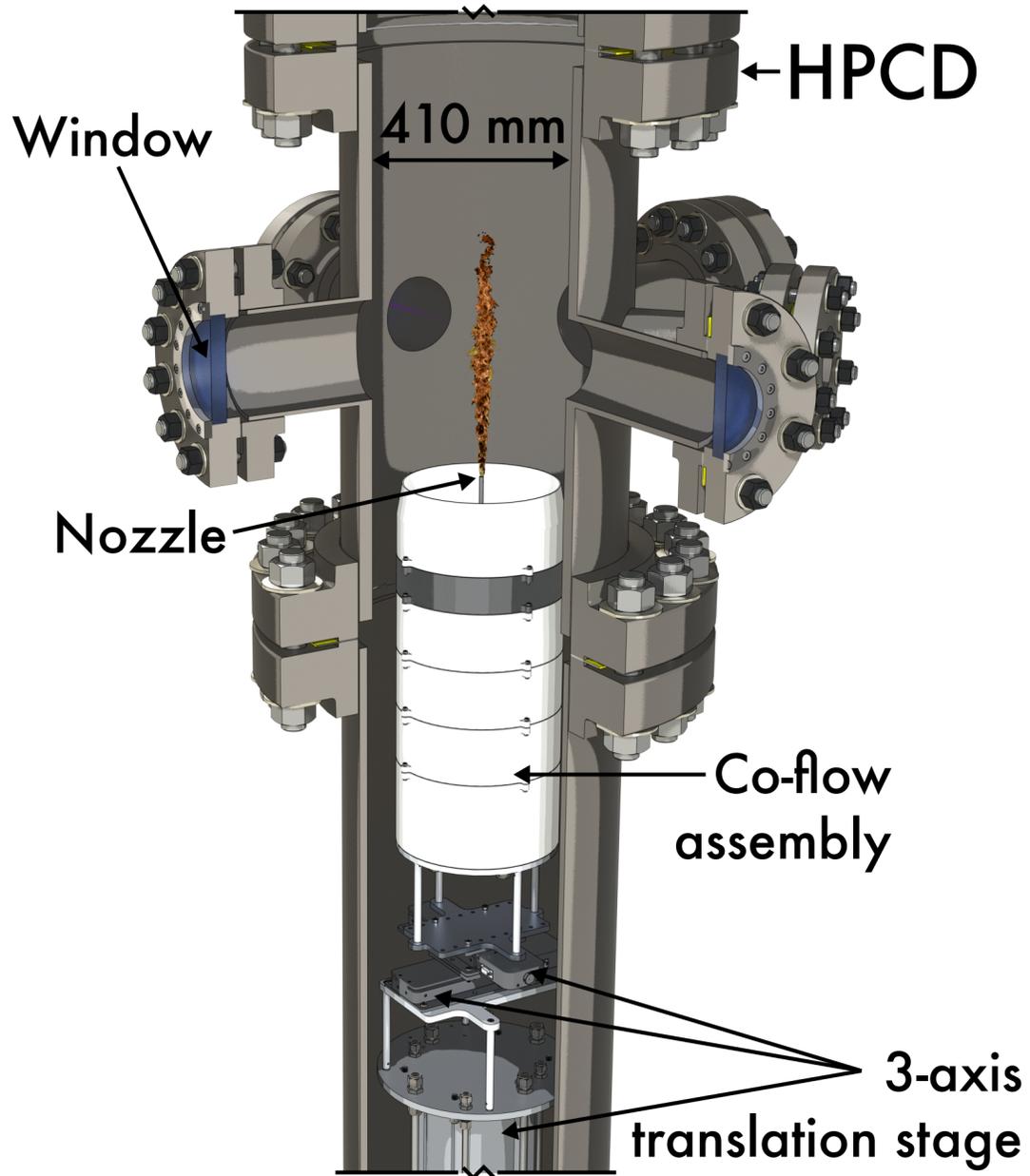
The flame edge is protected from the incoming flow

Detachment by turbulence-induced local extinction

Can we predict detachment quantitatively? *i.e.*, can we model detachment $U_d = f(x, y, \dots)$?



Flame detachment



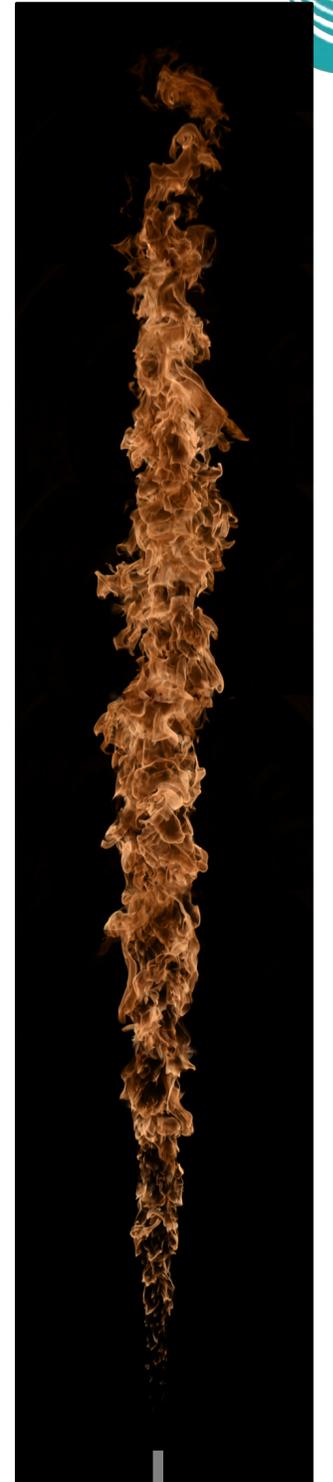
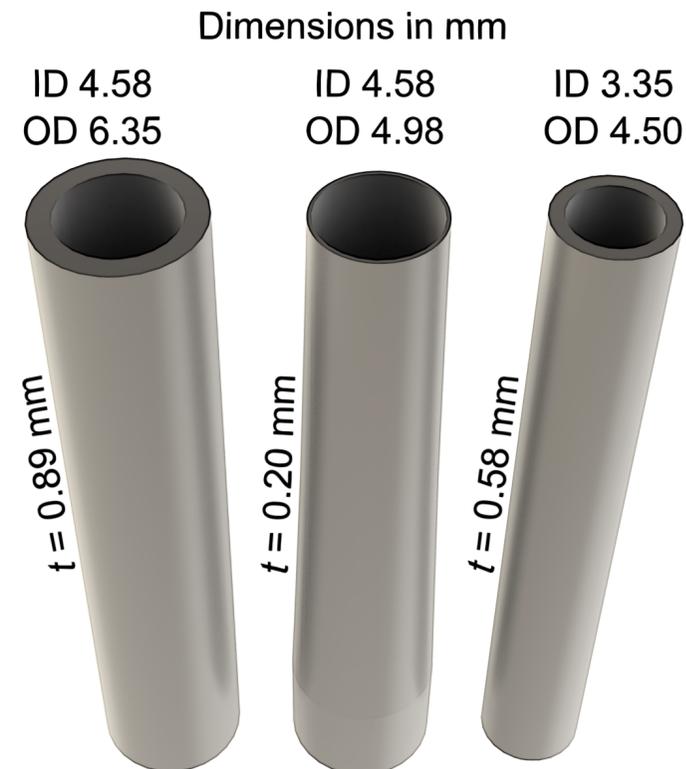
Fuels: methane & ethane

Pressures: $1 \leq P \leq 10$ bar

Jet velocities: $0.5 \leq U_j \leq 25$ m/s

Co-flow velocities: $0.3 \leq U_c \leq 0.9$ m/s

Reynolds numbers: up to 48,000



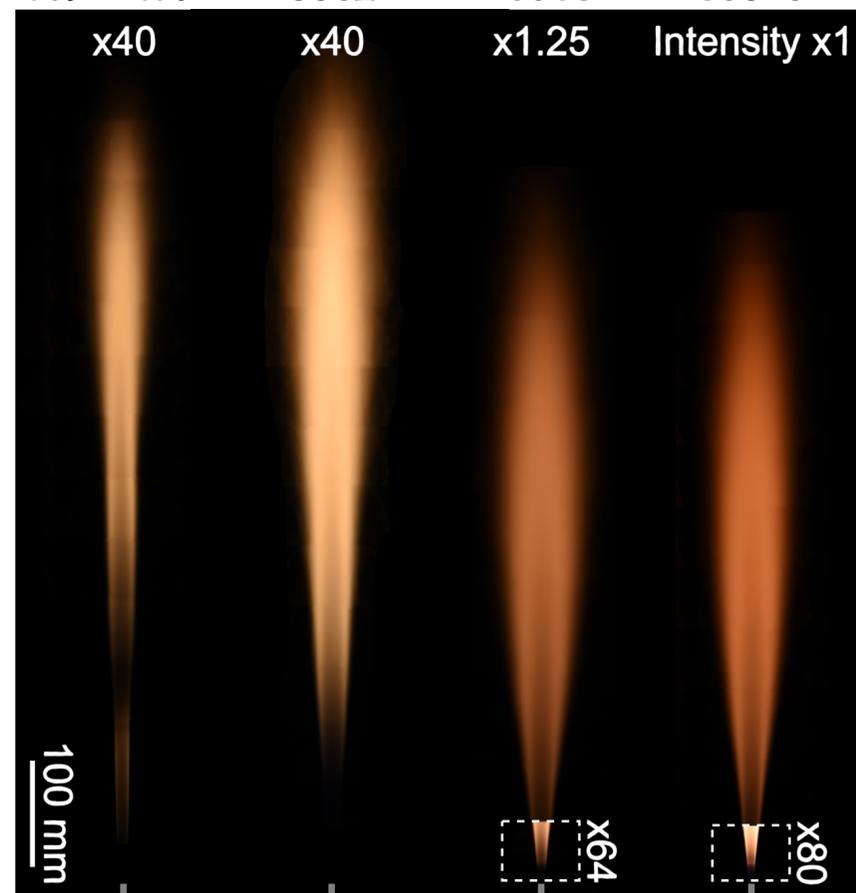
Instantaneous shots, stitched DSLR (6 bar)

Flame detachment



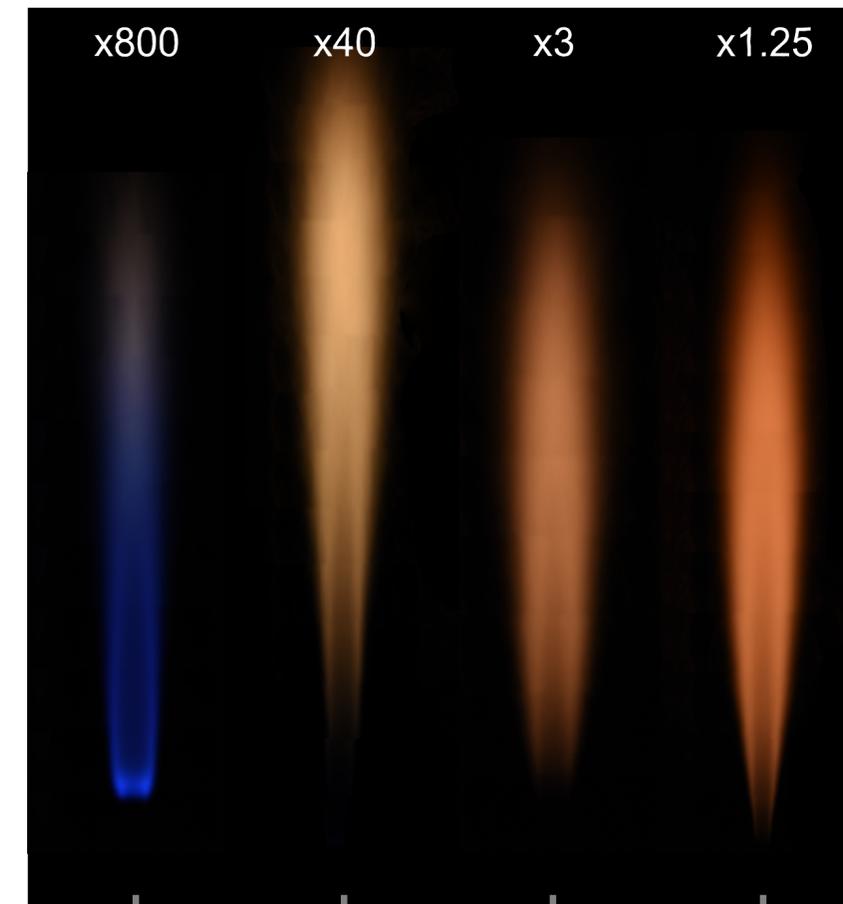
Attached

Methane	Ethane	Methane	Ethane
1 bar	1 bar	10 bar	6 bar
$U_j = 12 \text{ m.s}^{-1}$	10	7	10
$U_c = 0.6 \text{ m.s}^{-1}$	0.6	0.3	0.6
Re = 2397	5062	13985	30373



Lifted

Methane	Ethane	Methane	Ethane
1 bar	1 bar	7 bar	6 bar
$U_j = 24 \text{ m.s}^{-1}$	15	10	13
$U_c = 0.6 \text{ m.s}^{-1}$	0.6	0.3	0.6
Re = 4795	7593	13985	39485

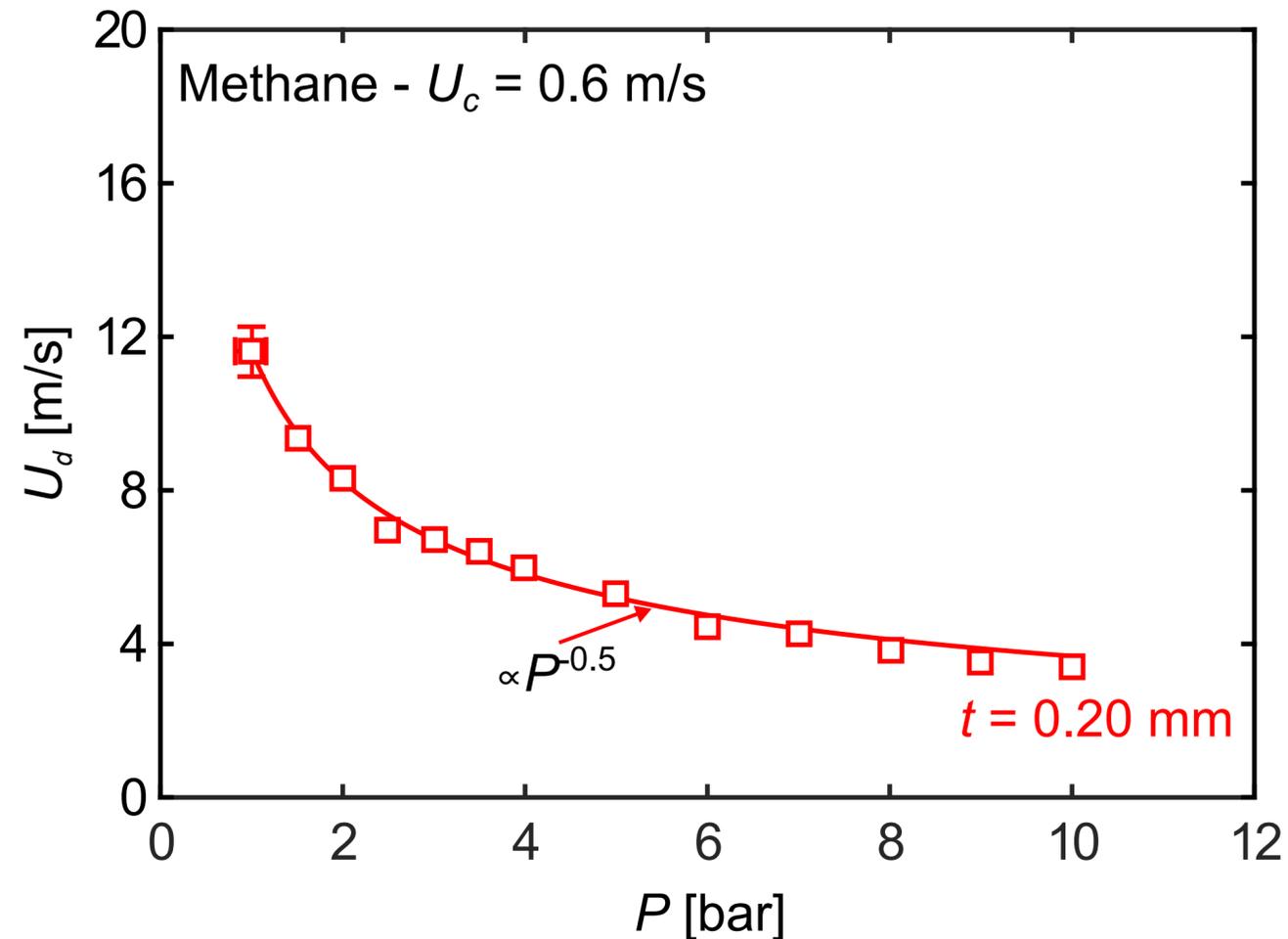


Starting with an attached flame, the jet velocity was increased progressively until detachment occurs. Detachment events were detected by eye



Flame detachment

Effect of pressure



If $t = 0.20$ mm, U_d decreases monotonically with P
It scales with $P^{-0.5}$

For methane, the laminar burning velocity S_L also
scales with $P^{-0.5}$

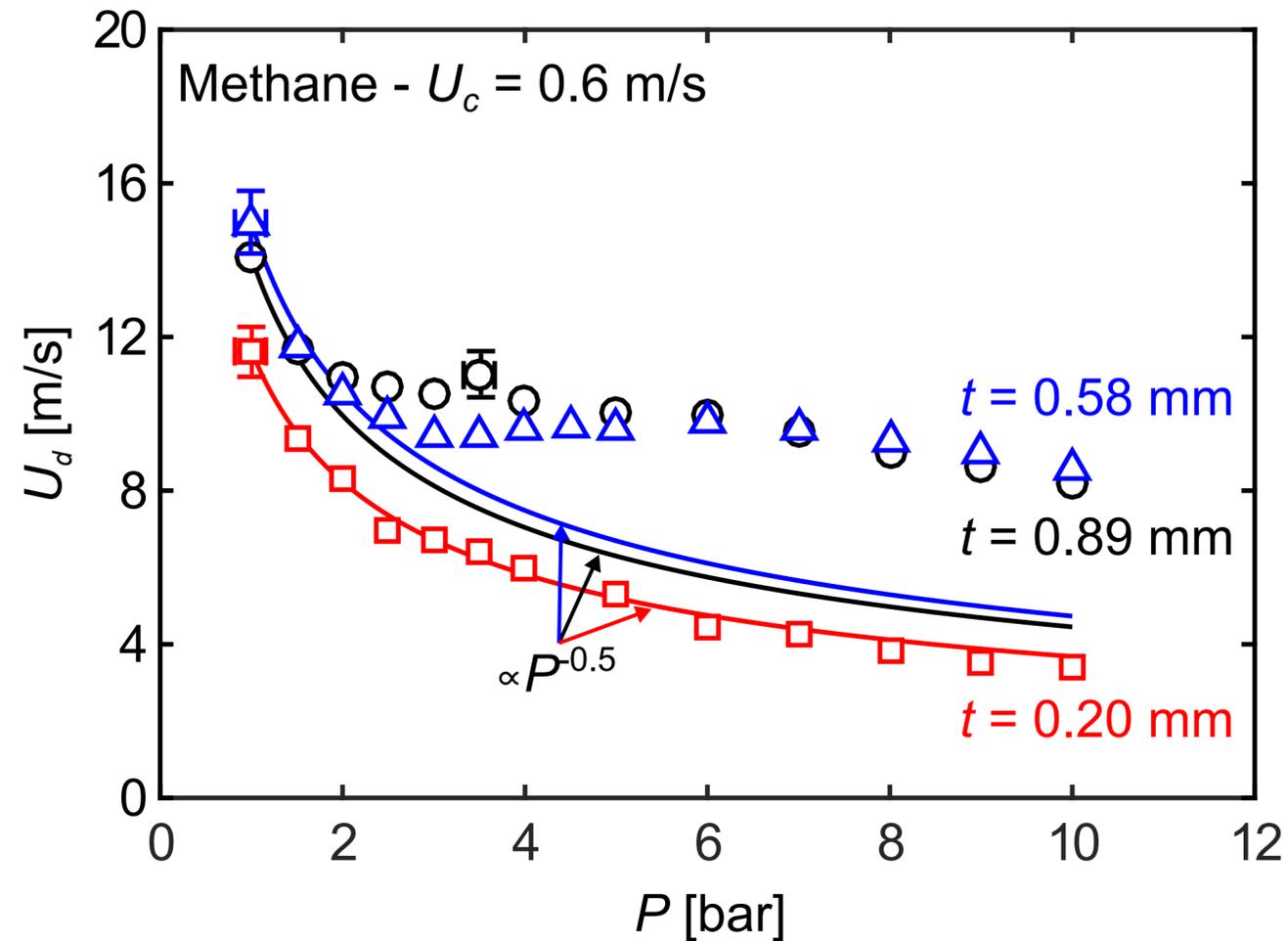
➔ If $t = 0.20$ mm, U_d and S_L are proportional

These experimental observations are consistent with aerodynamic detachment



Flame detachment

Effect of the nozzle thickness



Regardless of t , there is a critical P below which U_d scales with $P^{-0.5}$

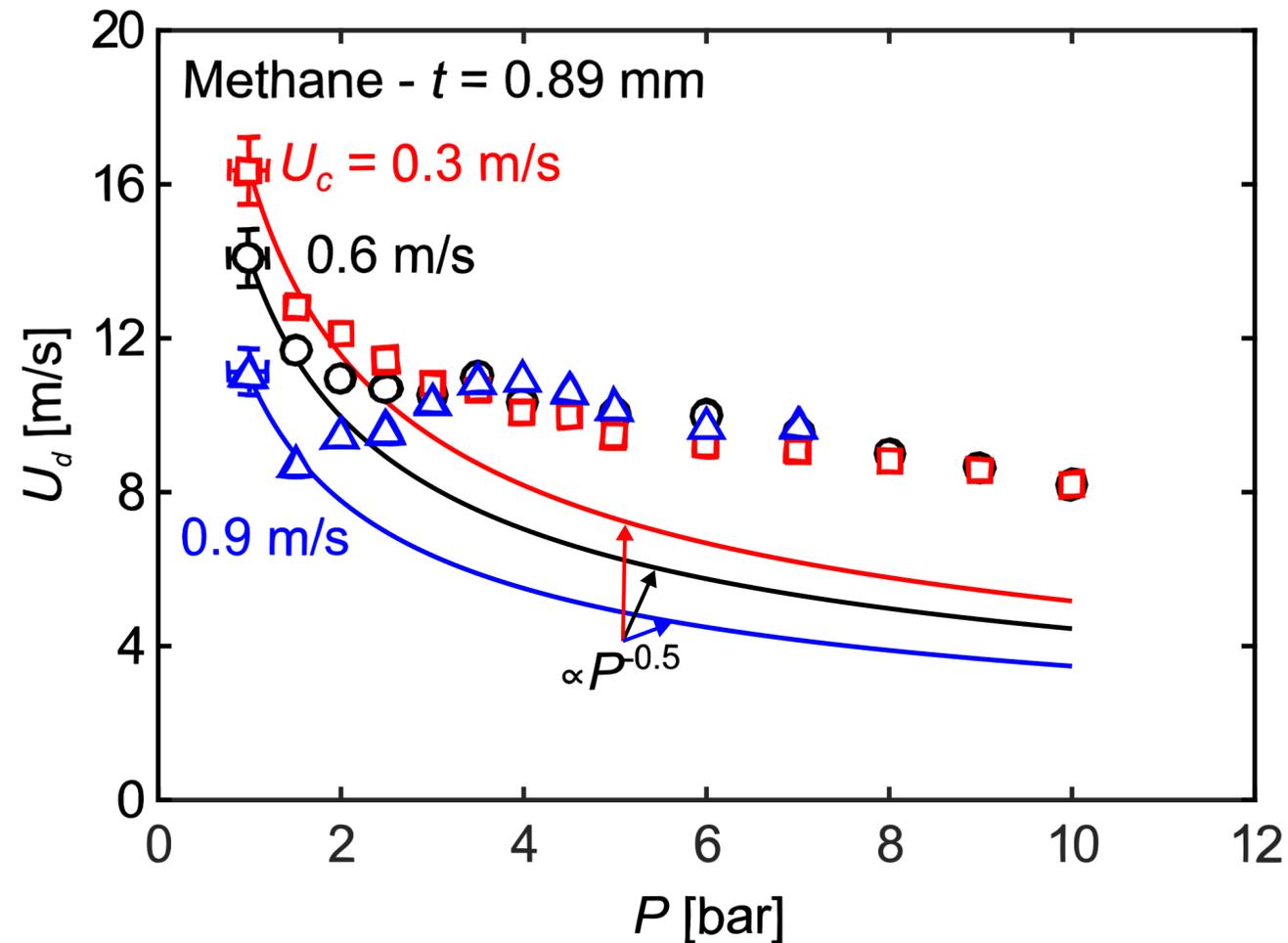
If $t = 0.58$ or 0.89 mm, a non-monotonic behavior is observed

➔ This suggests that the detachment mechanism changes as pressure increases



Flame detachment

Effect of the co-flow



If $t = 0.89$ mm and $P < 3$ bar, U_d is a function of U_c

➔ Aerodynamic detachment

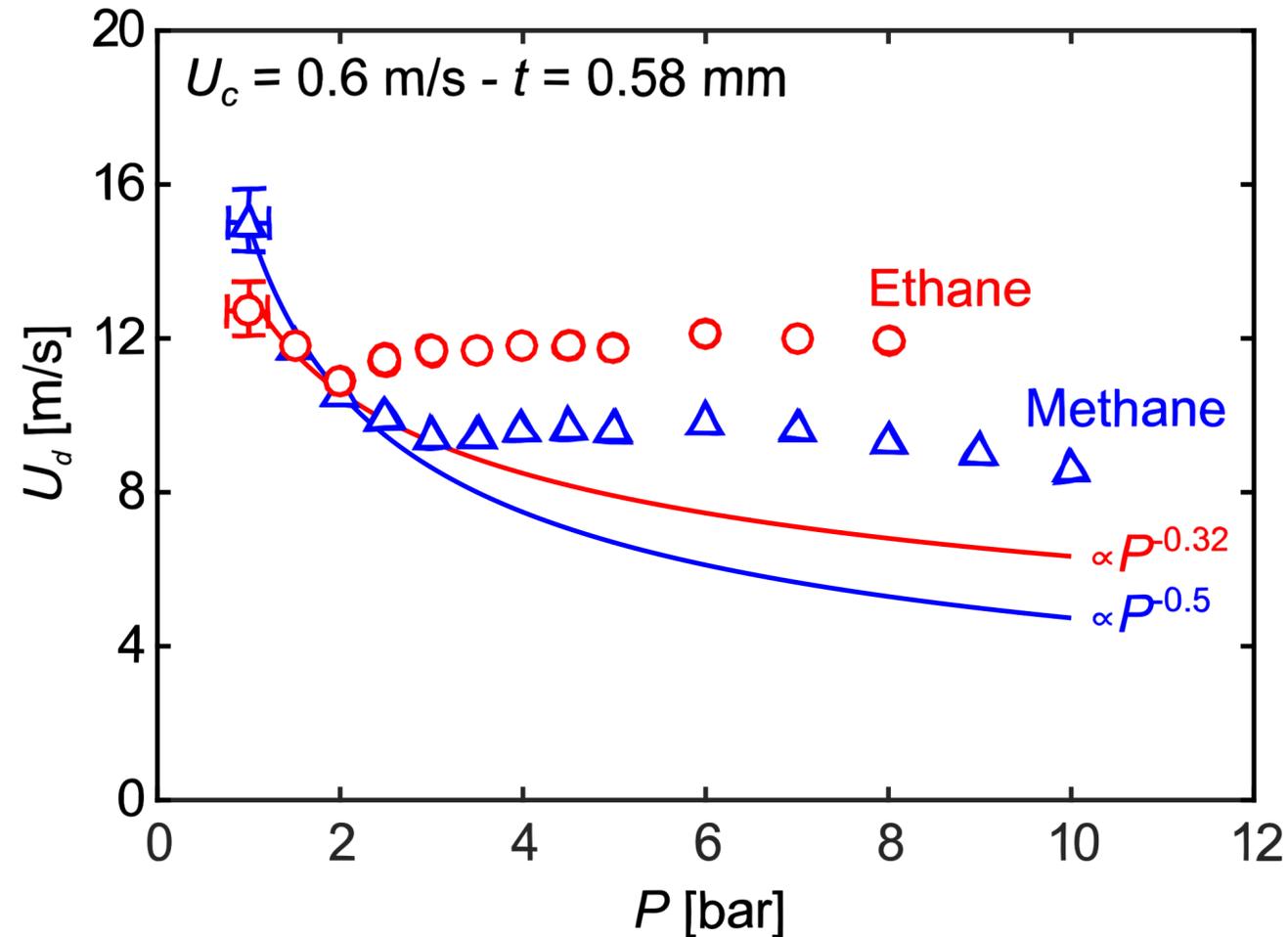
If $t = 0.89$ mm and $P > 3$ bar, U_d is not a function of U_c

➔ Detachment by local extinction?



Flame detachment

Effect of the fuel

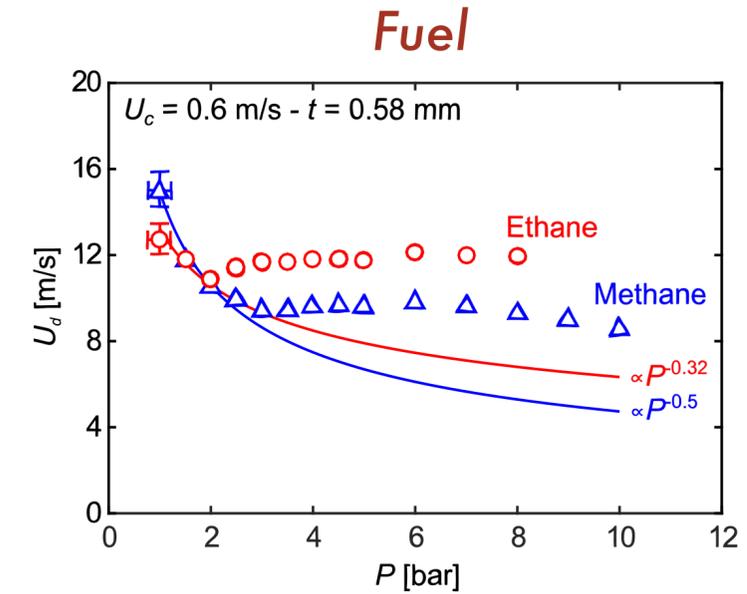
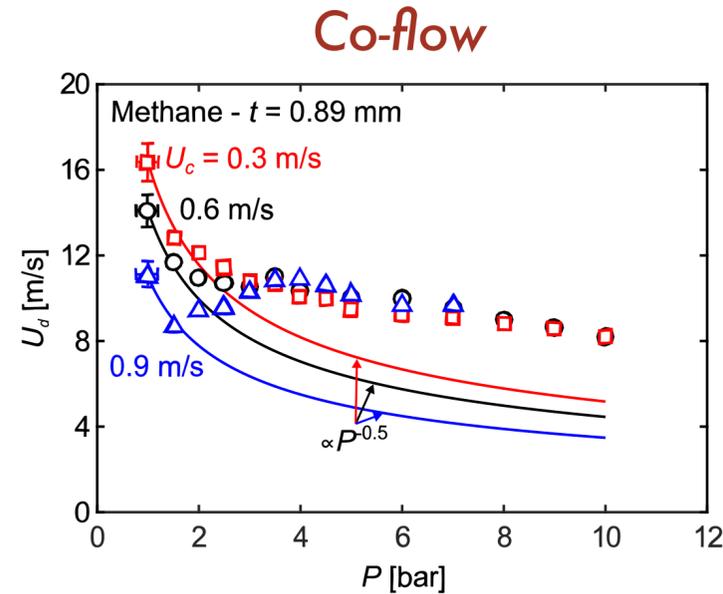
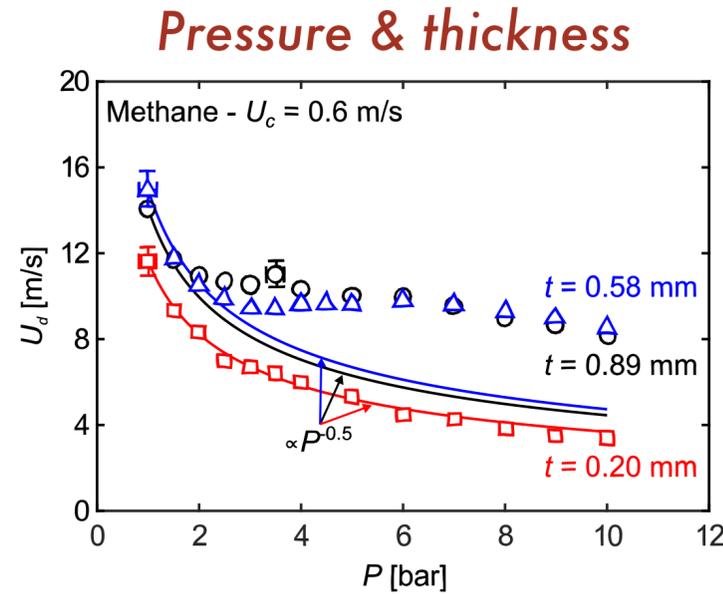


For ethane, if $P < 2$ bar, U_d is also proportional to S_L for ethane because S_L scales with $P^{-0.32}$

If $P > 2$ bar, ethane jet flames are more resistant to detachment



Flame detachment



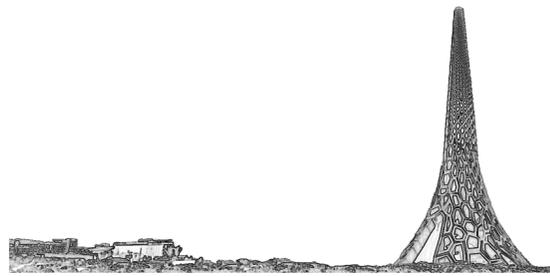
Increasing pressure allows to continuously cross boundaries between detachment mechanisms

➔ This is useful to develop models and predictive tools

This is "harder" to achieve with fuel or nozzle thickness

Effects of the mixture fraction

Boring to test > 10 nozzles



Flame detachment



$$t < 3\delta$$

Aerodynamic detachment

$$U_d = S_L / U_c \times f(\xi_{st})$$

ξ mixture fraction

The detachment velocity can be predicted for any fuel and pressure as long as it is known at one pressure and $t < 3\delta$

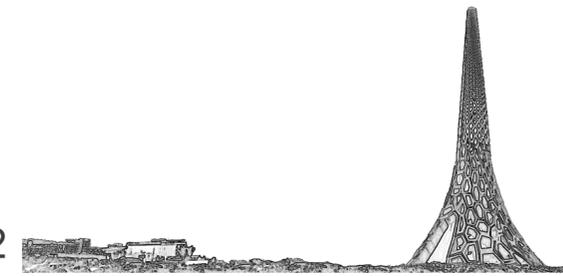
$$t = 3\delta$$

$$t > 3\delta$$

U_d does not depend on t and U_c

But U_d cannot be predicted yet...

From previous work at 1 bar, detachment is triggered by localized flame extinction



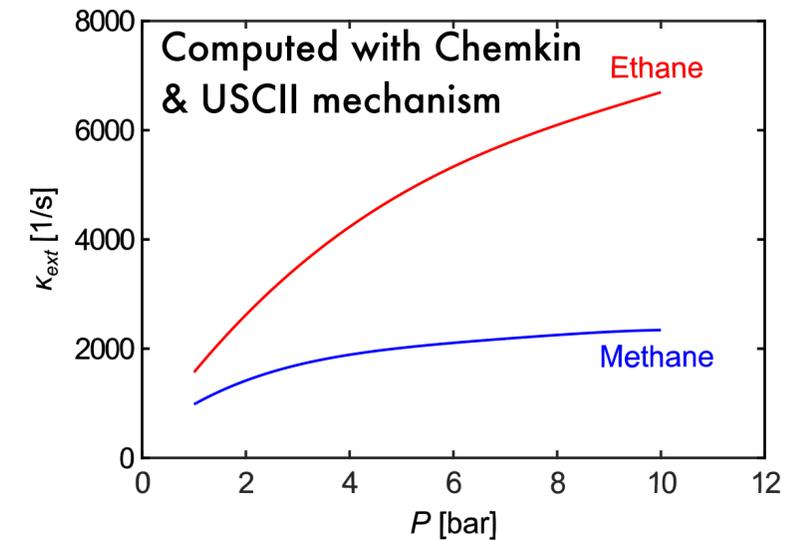
Flame detachment



Extinction of a flame front occurs when the flow time scale τ_f becomes smaller than the chemical time scale τ_c

The chemical time scale τ_c can be approximated here as the inverse of the extinction strain rate

For non-premixed flames, the flow time scale controlling local extinction is the fastest, *i.e.*, the Kolmogorov time scale of turbulence τ_η

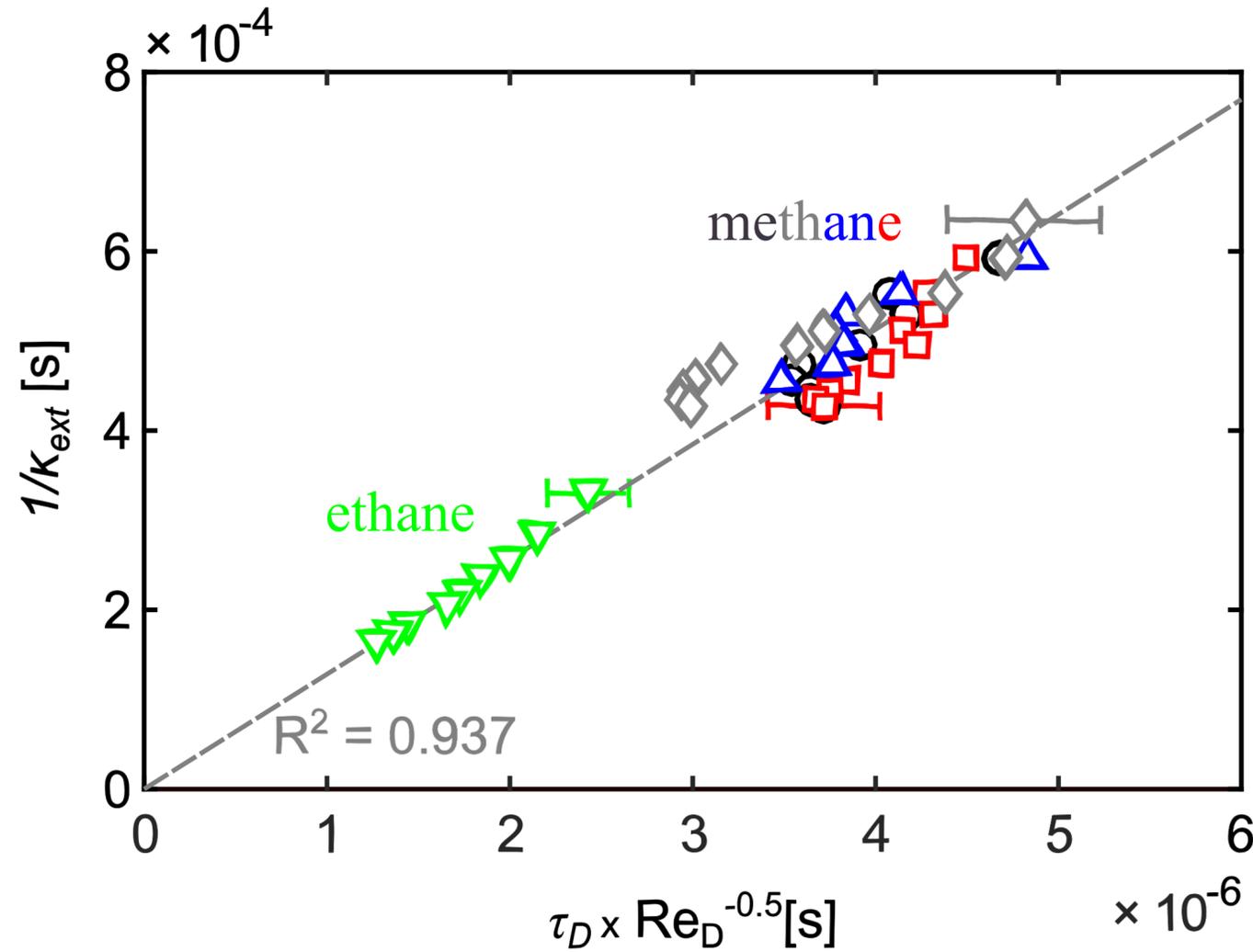


$$\tau_\eta \propto \tau_D \text{Re}_D^{-0.5}$$
$$\tau_D = \frac{D}{U_j} \quad \text{Re}_D = \frac{U_j D}{\nu}$$

assumes homogeneous isotropic turbulence

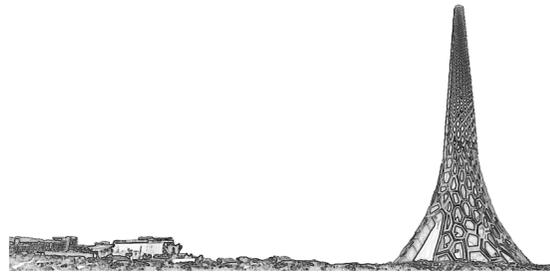


Flame detachment



$$U_D = 128 D Re_D^{-0.5} \kappa_{ext}$$

Assuming that the extinction strain rate can be computed accurately for other fuels, the detachment velocity should now be predictable also if $t > 3\delta$



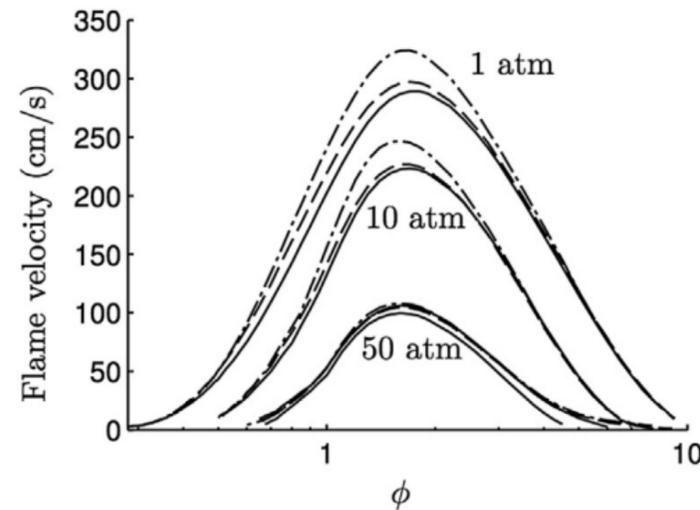
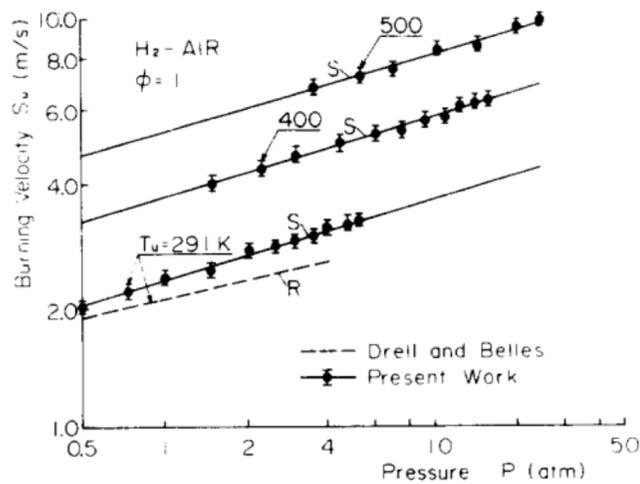
Flame detachment for H₂



$$t < 3\delta$$



$$U_d = S_L / U_c \times f(\xi_{st})$$

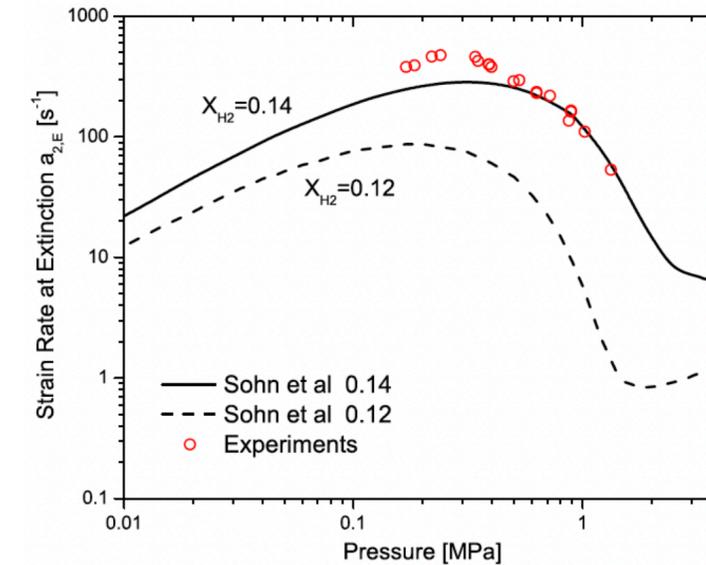


$$t = 3\delta$$

$$t > 3\delta$$



$$U_D = 128 D Re_D^{-0.5} \kappa_{ext}$$



Iijima & Takeno *Combust. Flame* 1986(65) Sanchez & Williams *Prog. Energy Combust. Sci.* 2014(41)

Niemann et al. *Proc. Combust. Inst* 2013(34)

At first, increasing pressure will retard detachment for H₂
 But ξ_{st} is closer to the fast fuel jet for H₂, which promotes detachment

H₂ flames are very resistant to strain-induced extinction
 Above some moderate pressure, detachment will be promoted for H₂



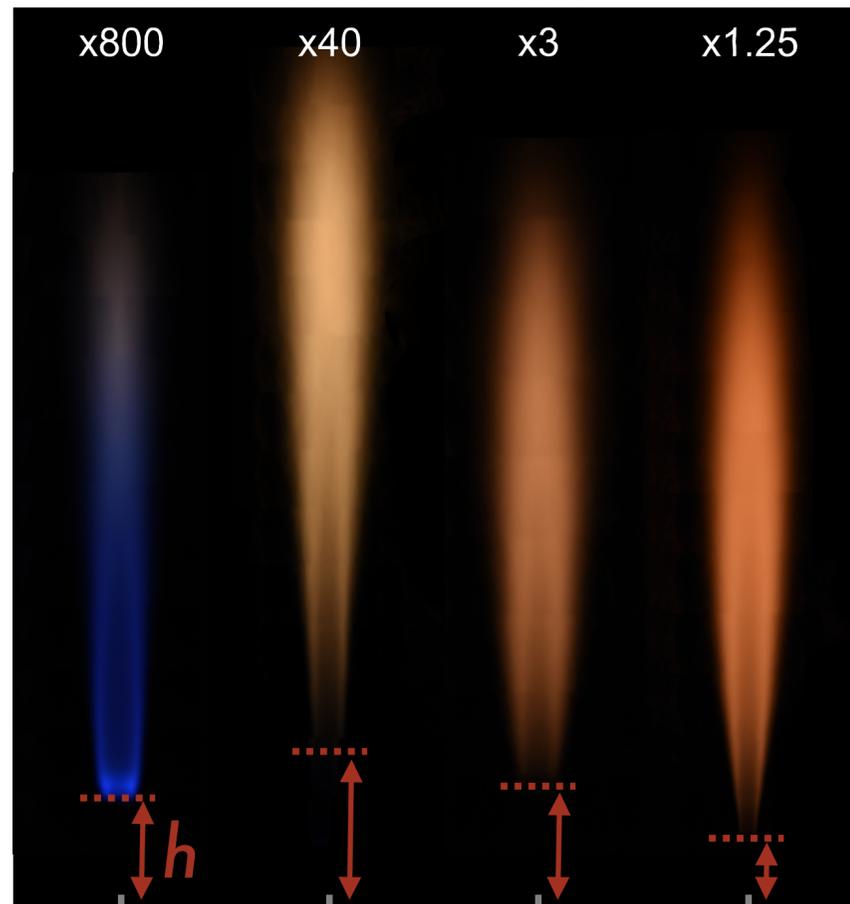
Lift-off height



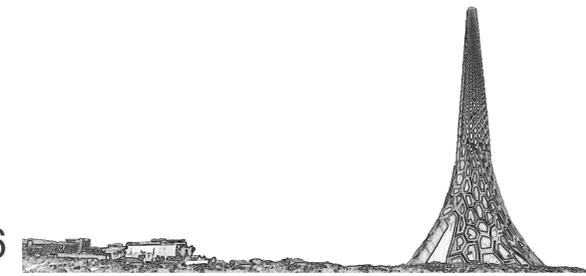
Once detached and lifted, how far away from the nozzle is the flame base?

Lifted

Methane 1 bar $U_j = 24 \text{ m.s}^{-1}$ $U_c = 0.6 \text{ m.s}^{-1}$ Re = 4795	Ethane 1 bar 15 0.6 7593	Methane 7 bar 10 0.3 13985	Ethane 6 bar 13 0.6 39485
x800	x40	x3	x1.25

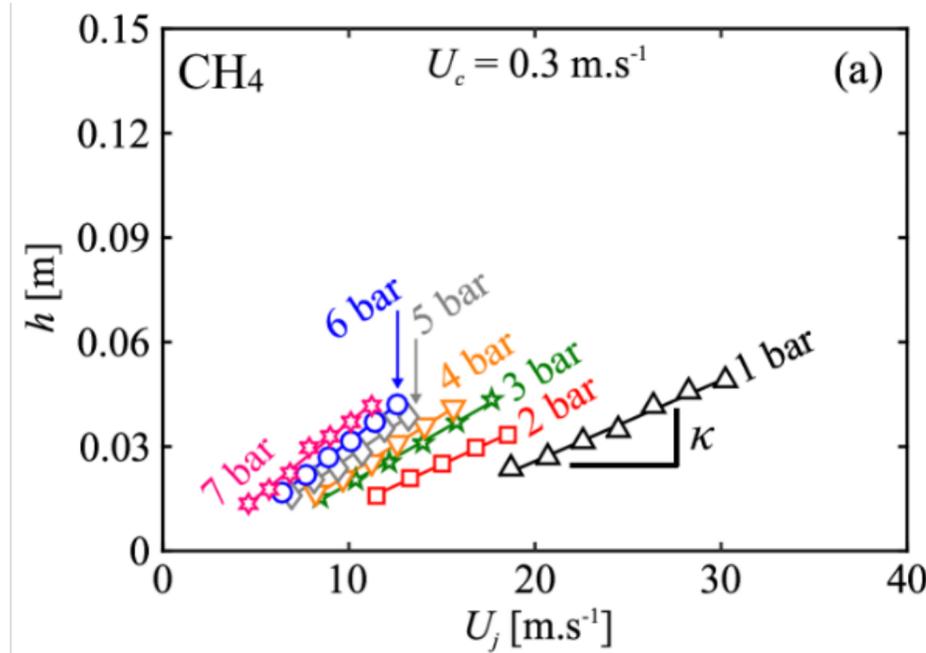


Where are the conditions met for the leading edge of the flame to stabilize?





Lift-off height



Regardless of pressure, the h vs. U_j curves are linear

The lift-off height increases with the bulk jet velocity

The lift-off height increases with pressure

The slope K is positive and is not too sensitive to pressure

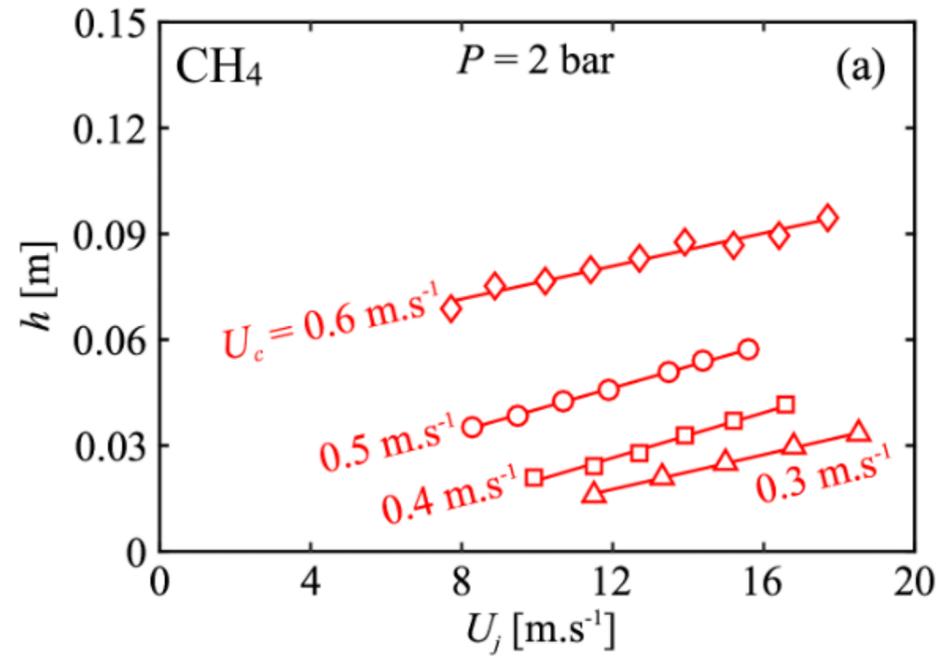
If the co-flow velocity is increased to $U_c = 0.6 \text{ m/s}$:

➔ The slope K decreases with pressure and can be negative





Lift-off height



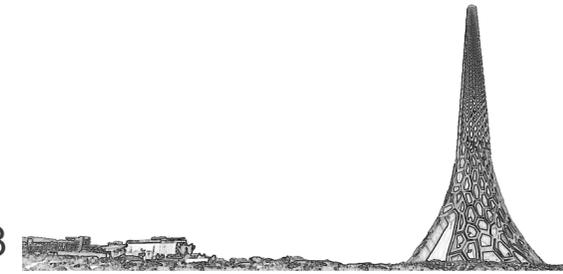
The lift-off height increases with the co-flow velocity

At 2 bar, slope K does not depend on the co-flow velocity

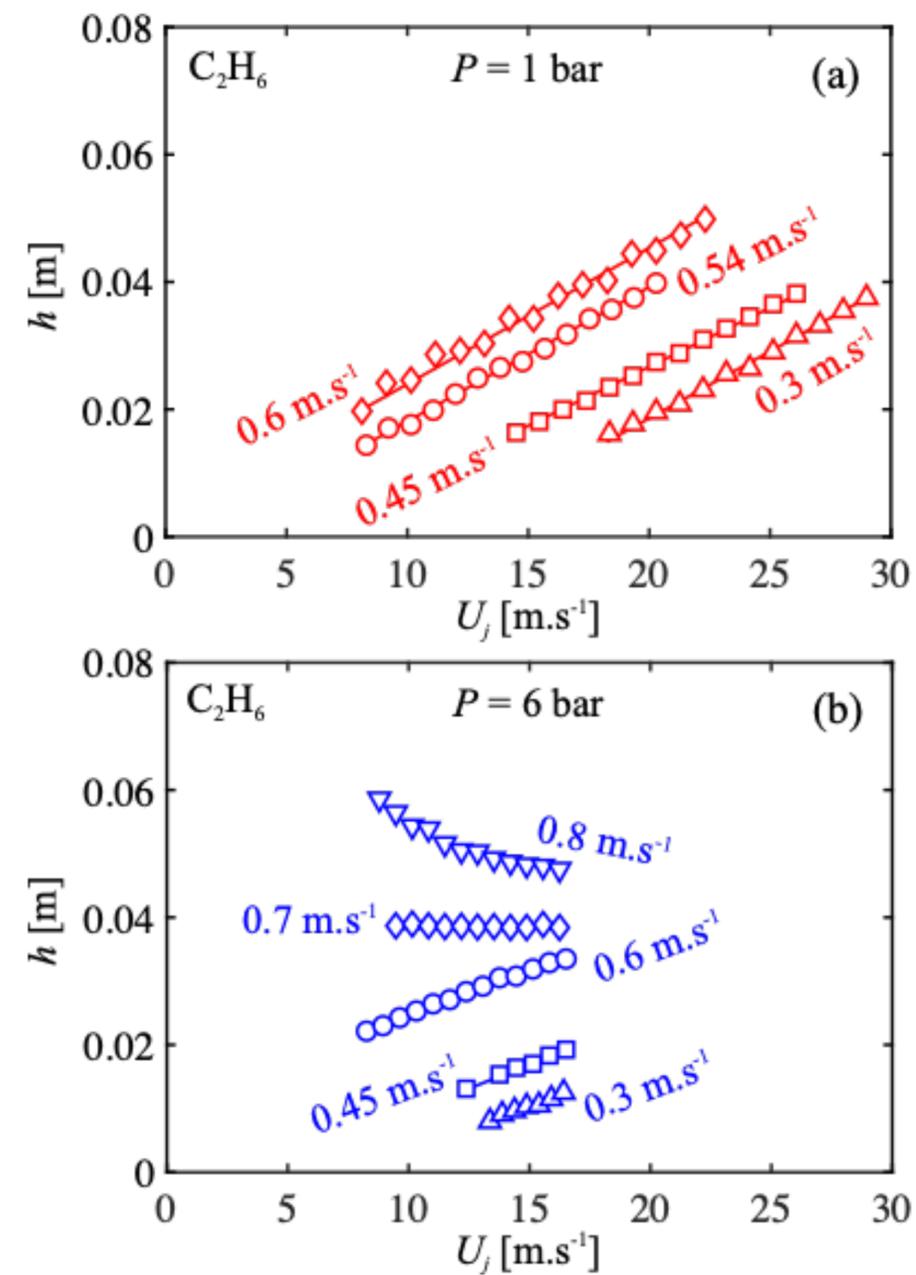
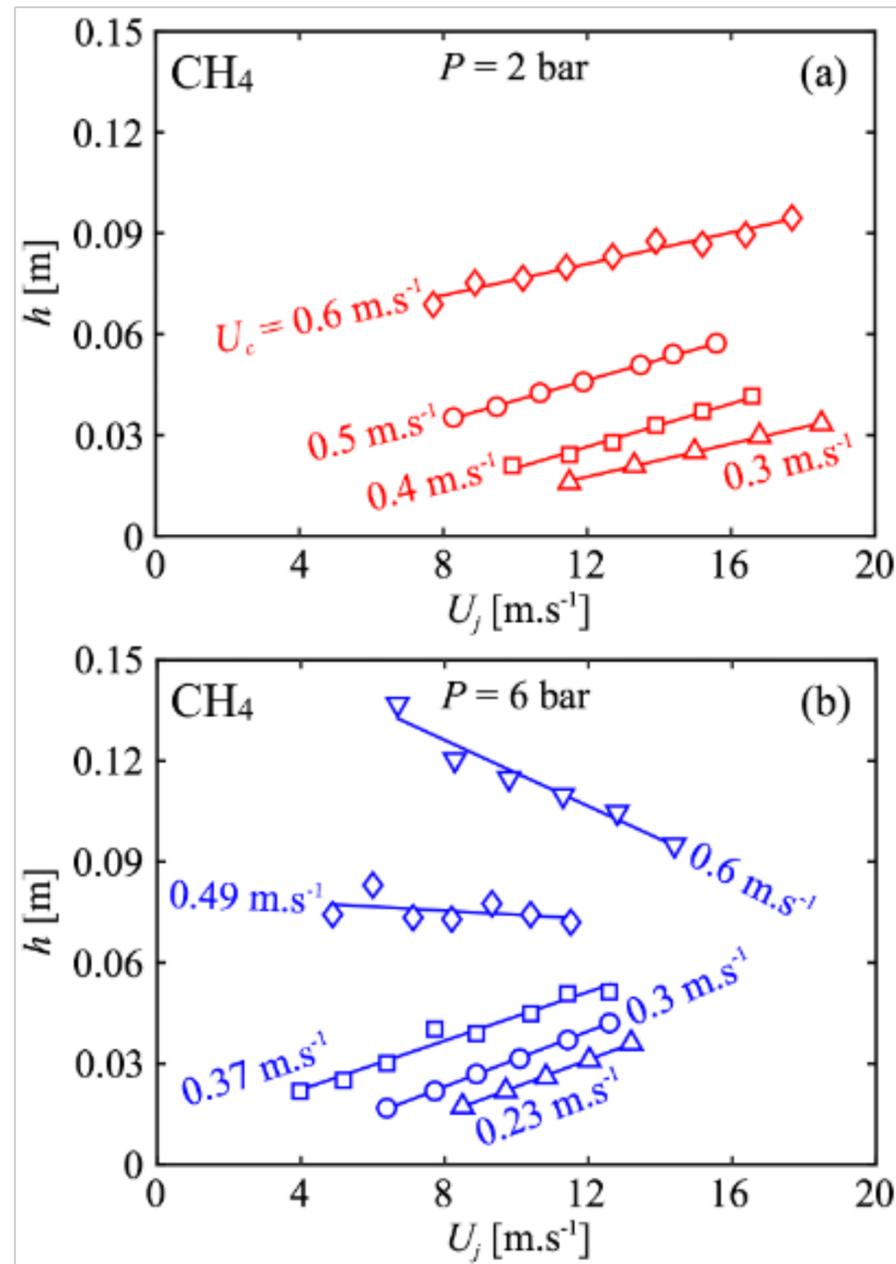
Further increasing the co-flow velocity leads to blow-off

If the co-flow velocity exceeds a critical value:

➔ The slope K decreases with the co-flow velocity and can be negative



Lift-off height

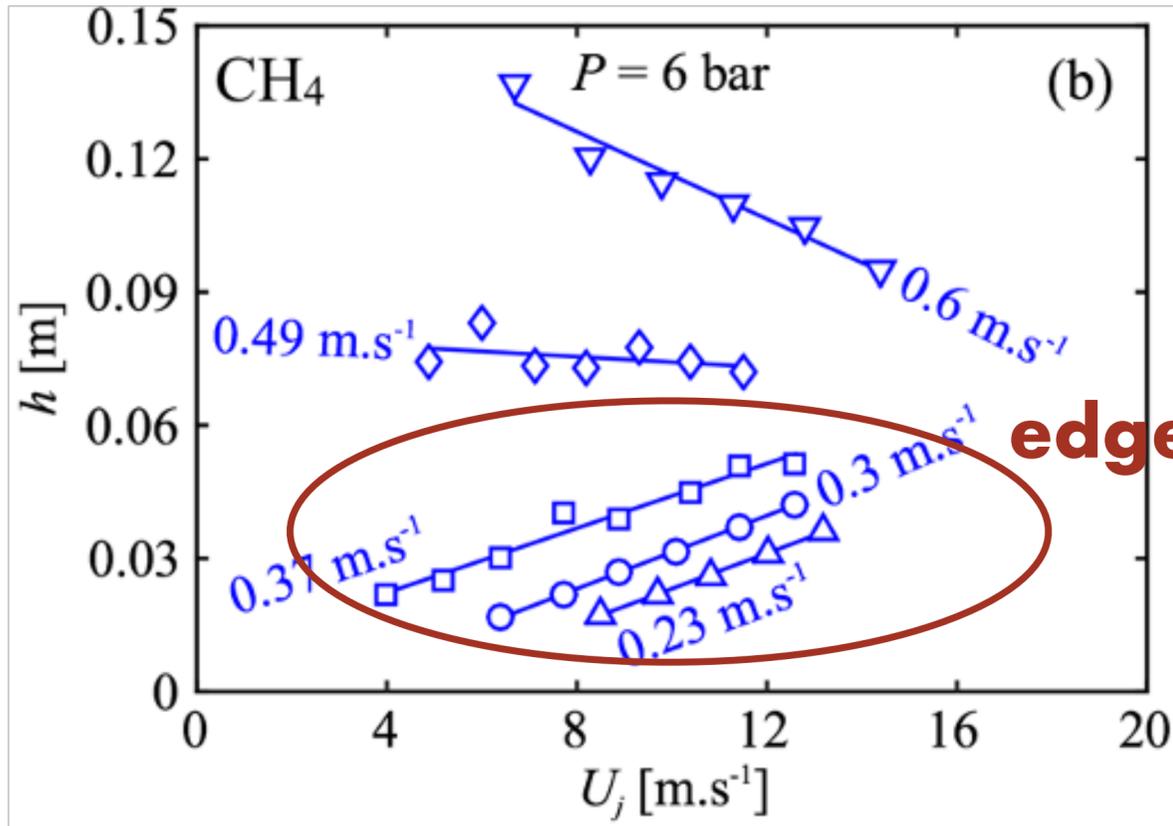


The same behavior is observed with ethane, albeit for a larger co-flow velocity

There seem to be a transition in the flame's stabilization mechanism if both pressure and co-flow velocity are large enough



Stabilization mechanisms



edge flames

- non-premixed flame sheet
- sits on a stoichiometric contour
- counters the incoming flow at a velocity close to the laminar burning velocity S_L

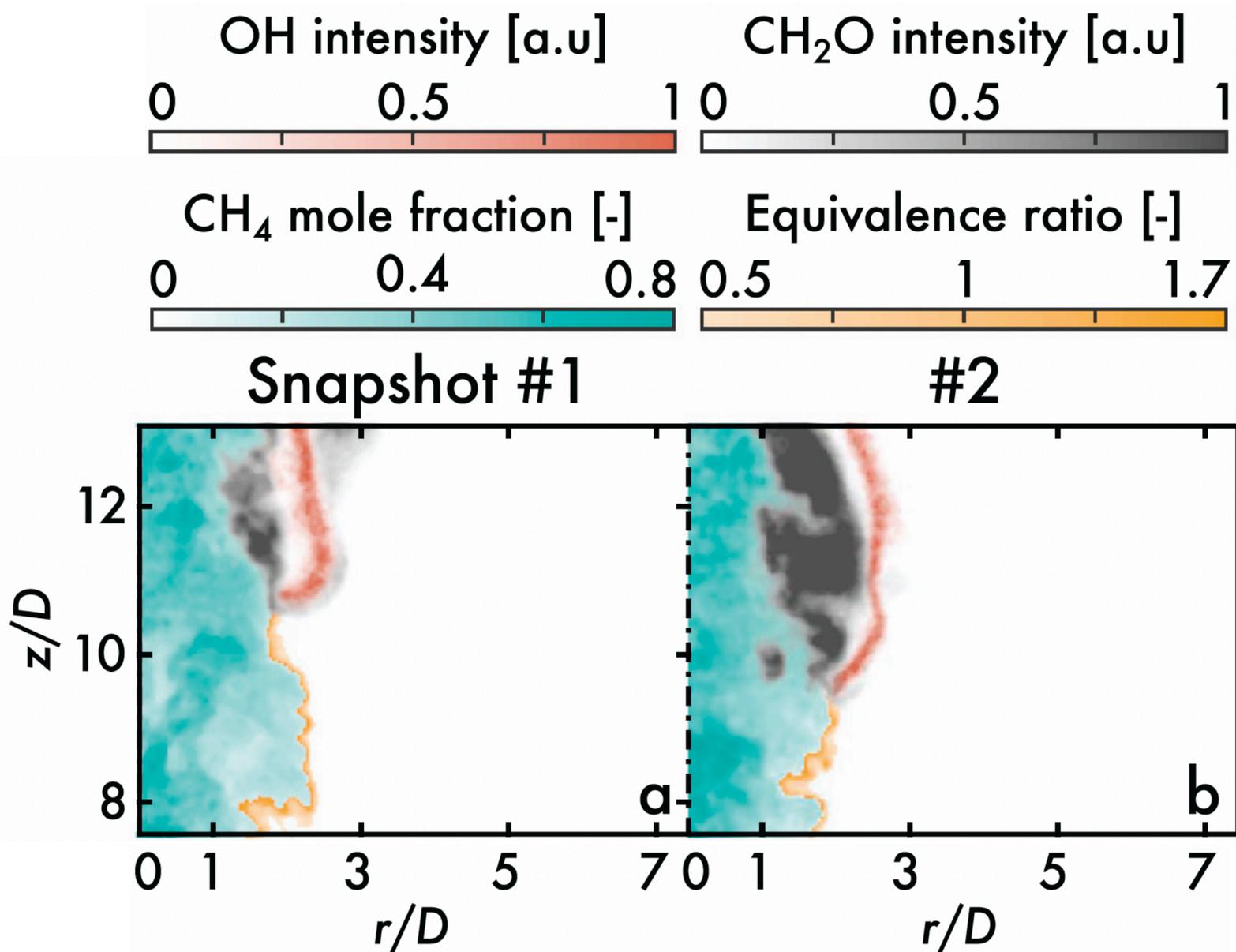
$$\phi = 1 \ \& \ 0 < u < 3S_L$$

Chung S.H Proc. Combust. Inst. 2007
 Li et al. Combust. Flame 2010
 Karami et al. J. Fluid Mech. 2015

More generally $0 < u < (\rho_u/\rho_b)^{1/2}S_L < 3S_L$ for air as the oxidiser



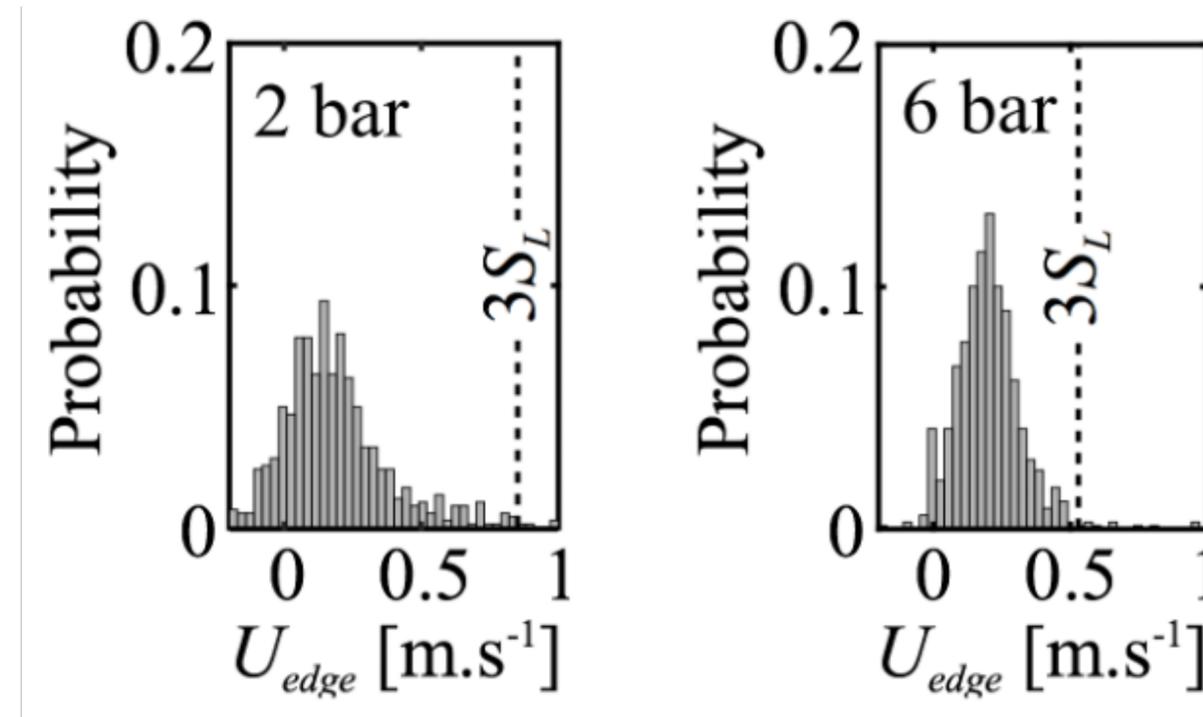
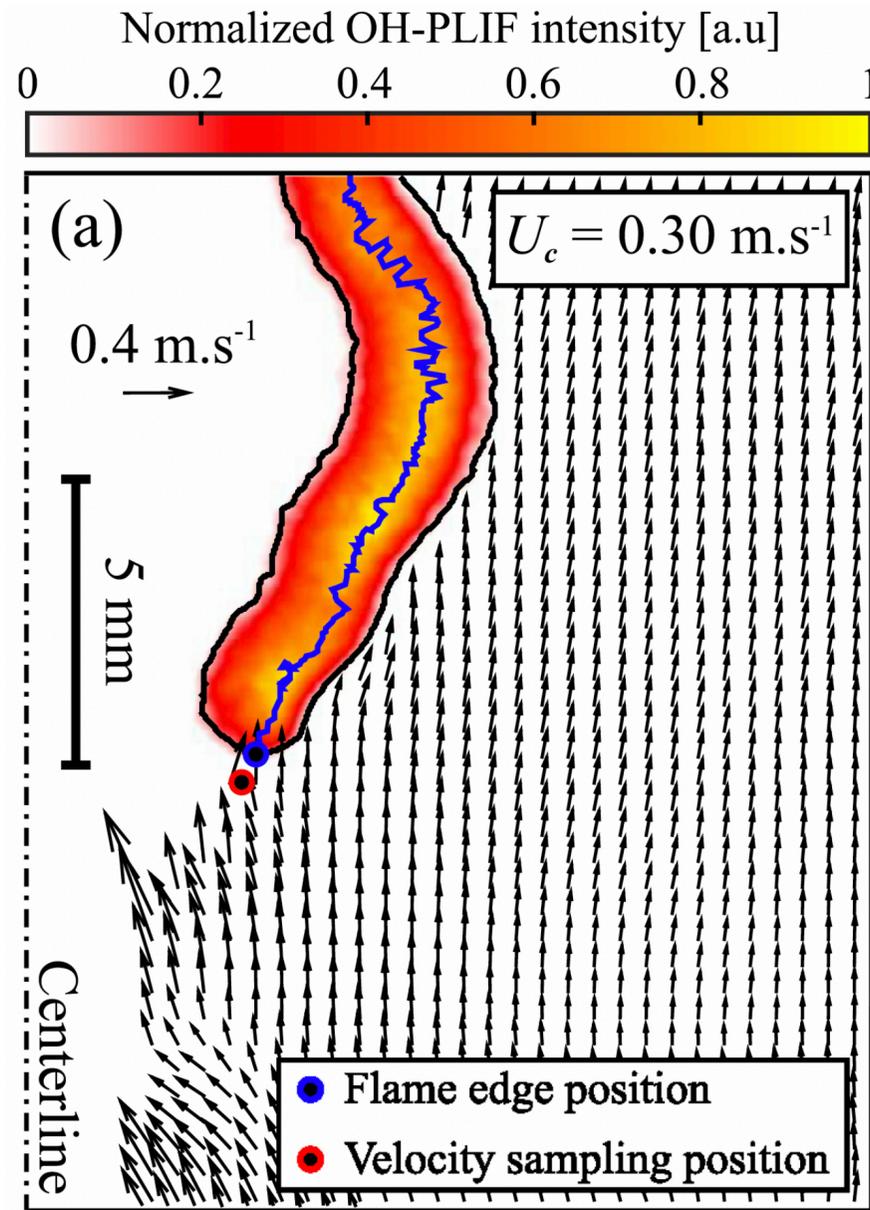
Stabilization mechanisms



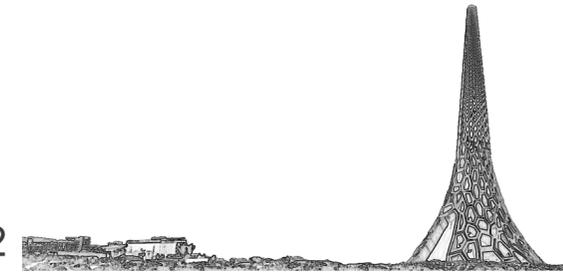
Multi-scalar imaging confirm the edge-flame structure at 6 bar and $U_c = 0.3$ m/s



Stabilization mechanisms



Combined velocimetry (PIV) and OH-PLIF confirm edge-flame stabilization



Stabilization mechanisms



Can we predict the lift-off height for wide ranges of fuel, pressure, co-flow, geometry, ...?

A non-dimensional lift-off height is defined

Kalghatgi et al. Combust. Sci. Technol. 1984

It can be predicted well by a physics-based model, which features quantities that can be easily computed

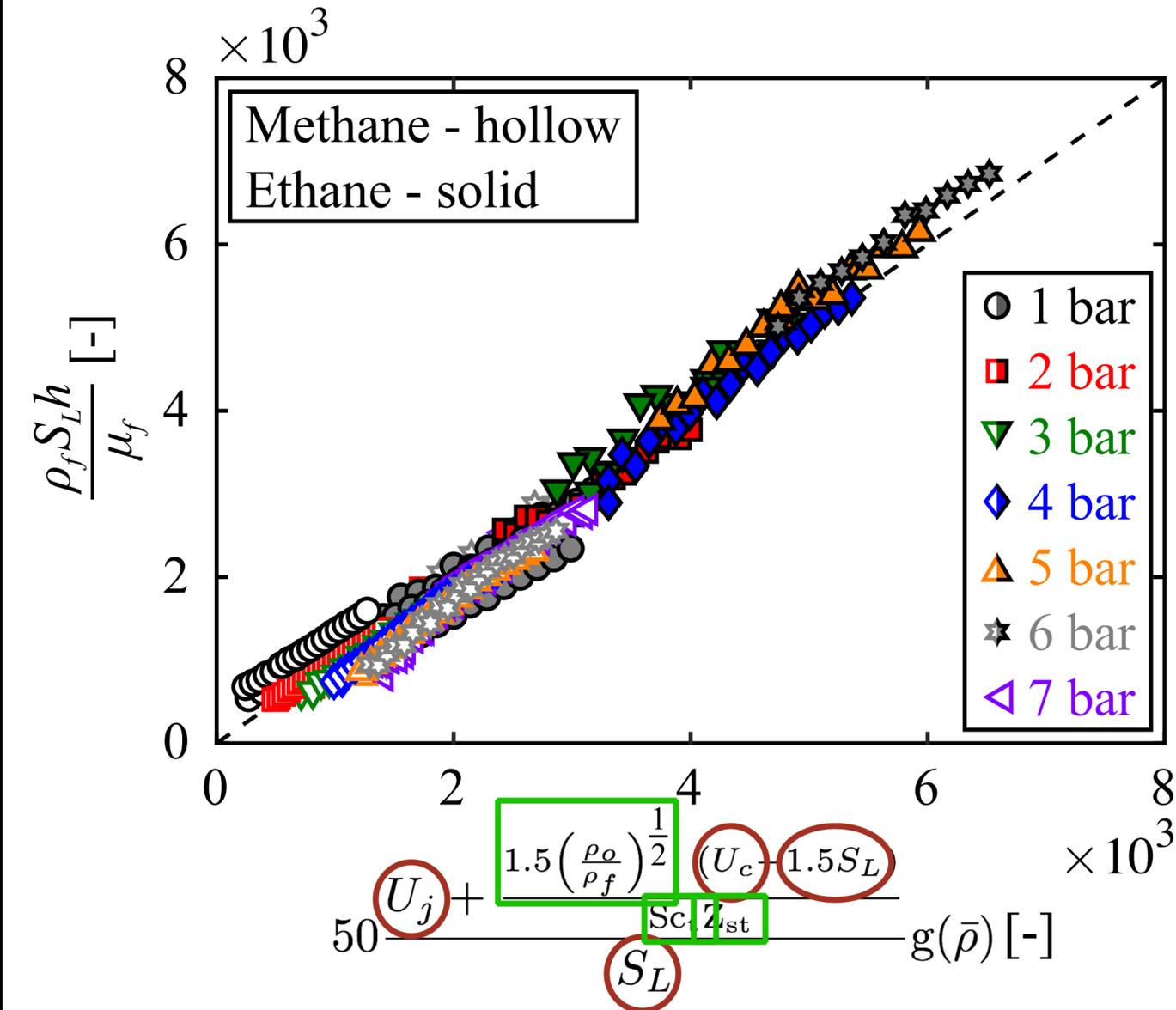
h increases if U_j increases

h decreases if S_L increases

The co-flow pushes the flame downstream

- *Mixing via Kelvin-Helmholtz instabilities depends on the density ratio*
- *The stoichiometric mixture fraction controls how influential the co-flow is*
- *A turbulent Schmidt number accounts for velocity and species spread rates*

A corrective term allows for a negative Y-intercept of the h vs. U_j curve

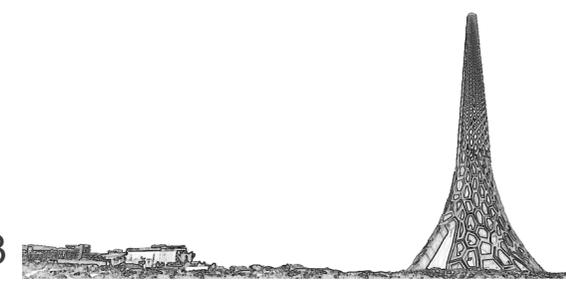


Upatnieks Combust. Flame 2004

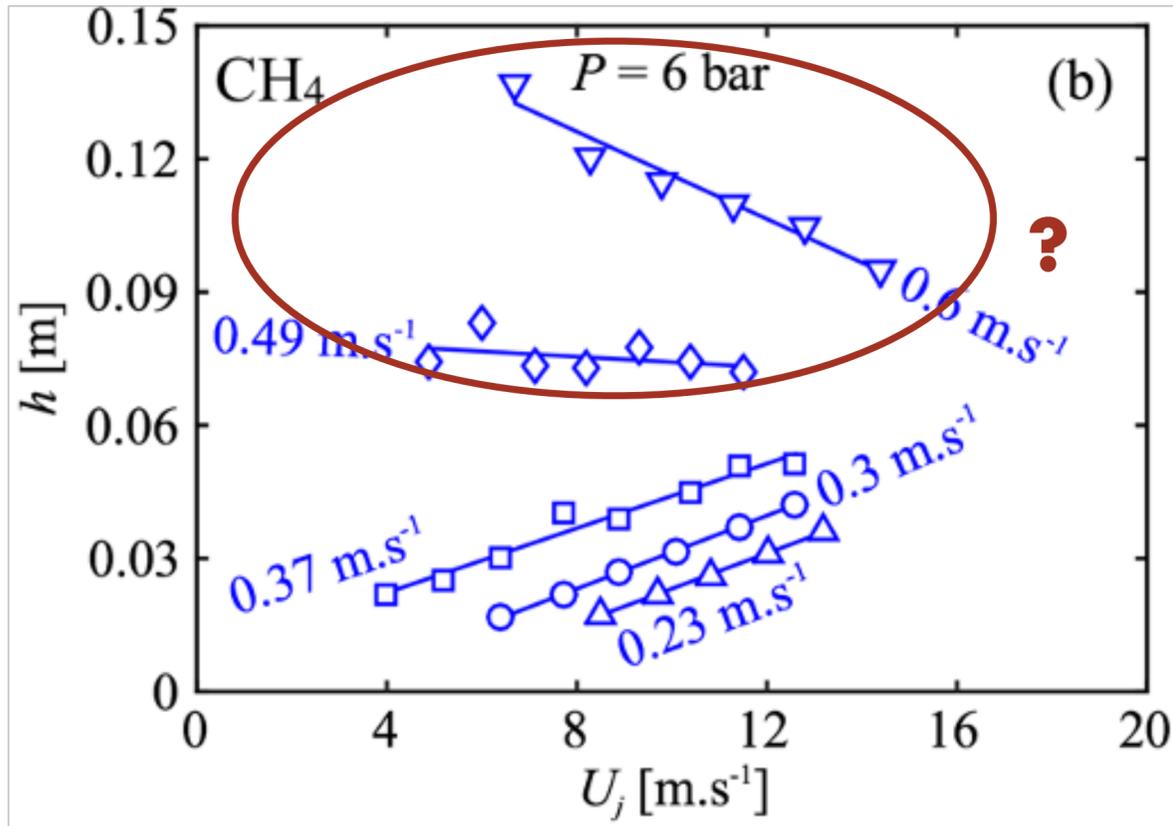
Han and Mungal Combust. Flame 2003

Montgomery et al. Proc. Combust. Inst. 1998

Kalghatgi Combust. Sci. Technol. 1984



Stabilization mechanisms



What is the critical co-flow velocity above which inversion occurs?

Fuel	Pressure (bar)	U_c^{crit} (K = 0) (m/s)	U_c^{crit}/S_L (-)
Methane	3	0,6	3
Methane	6	0,48	3,2
Methane	7	0,43	3,0
Ethane	6	0,70	3,2
Ethylene	1	2,0	3

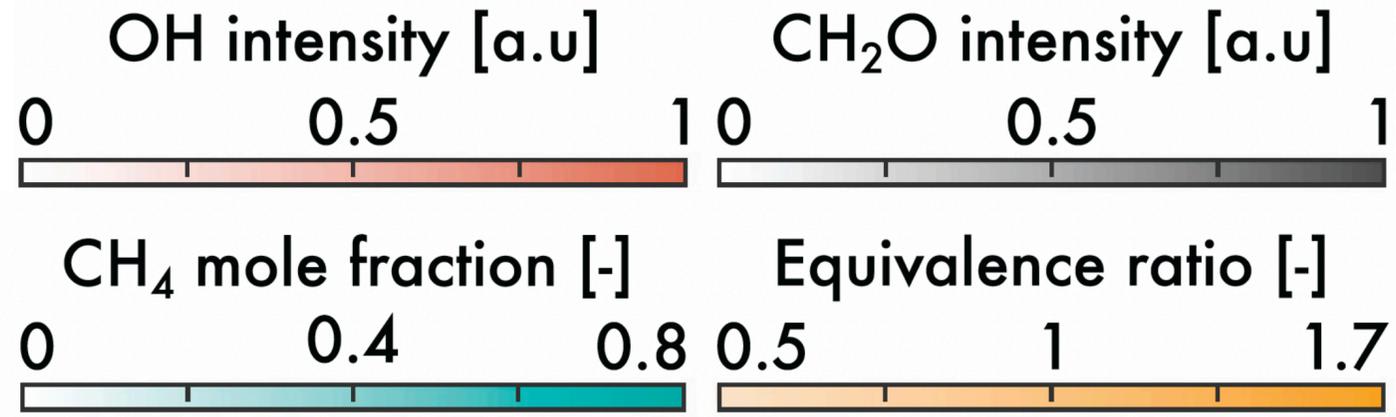
The critical co-flow velocity above which an edge flame cannot stabilize is $U_c^{crit} = 3S_L$

This is consistent with the experimental evidence of a maximum edge-flame speed of $3S_L$

What happens if $U_c > U_c^{crit} = 3S_L$?



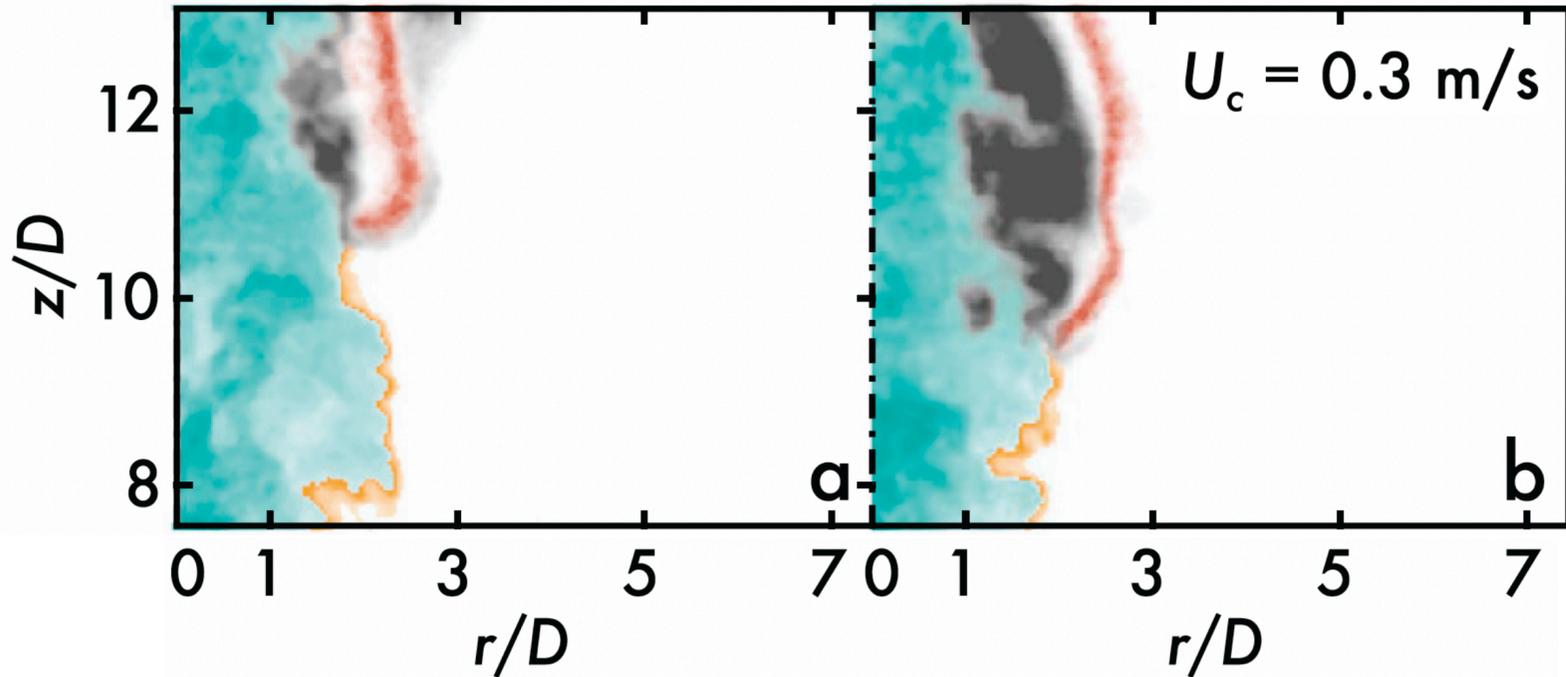
Stabilization mechanisms



Snapshot #1

#2

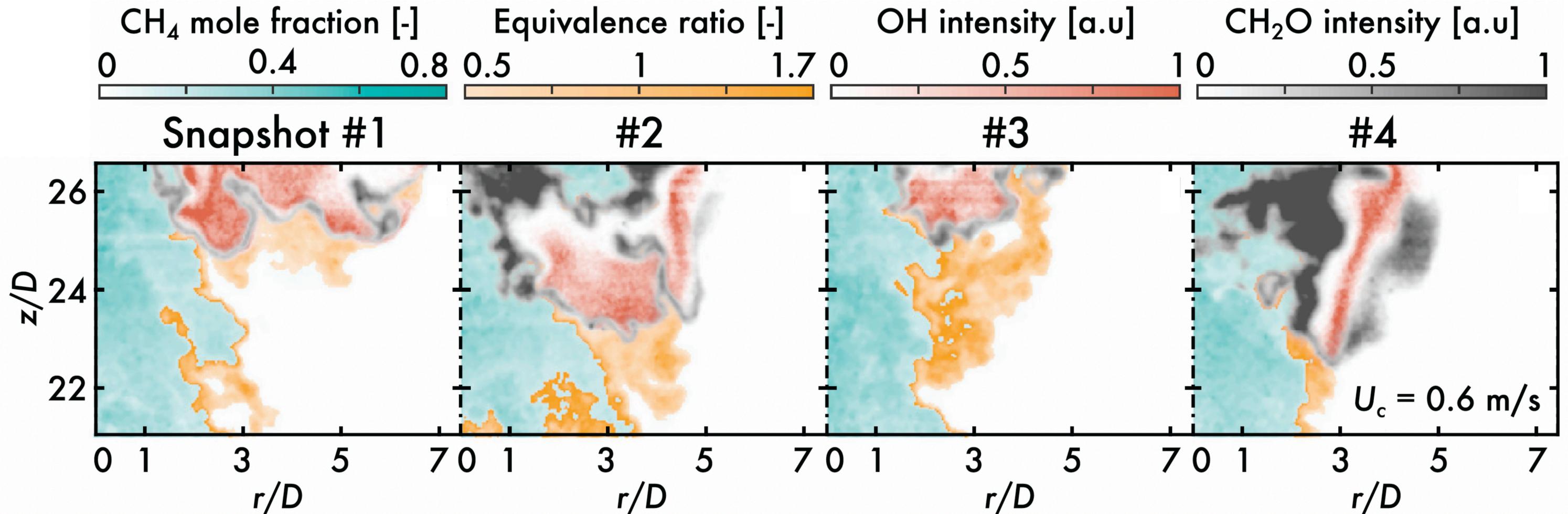
$U_c = 0.3$ m/s



Stabilization mechanisms



If the co-flow velocity is increased to $U_c = 0.6$ m/s ($U_{c,crit} = 0.5$ m/s):



➔ The flame is pushed further downstream, where the width of the flammable zone has increased significantly due to sufficient mixing

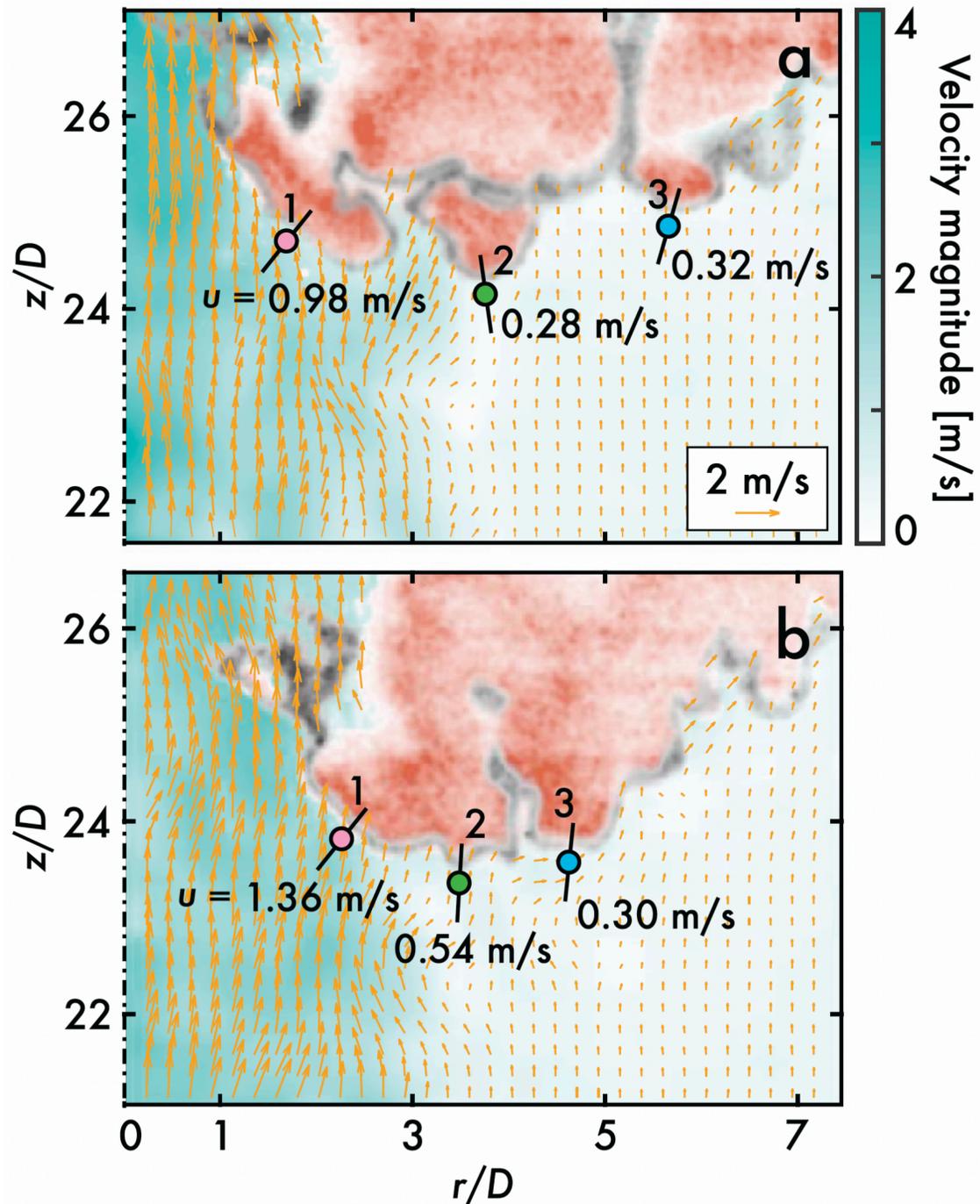
➔ The structure of the flame becomes vastly different



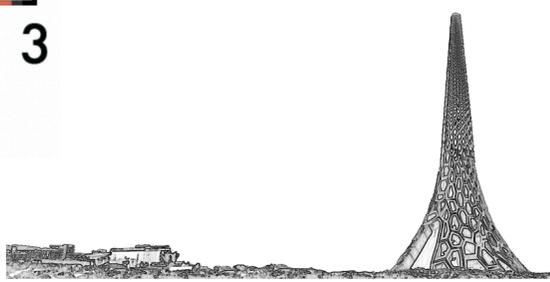
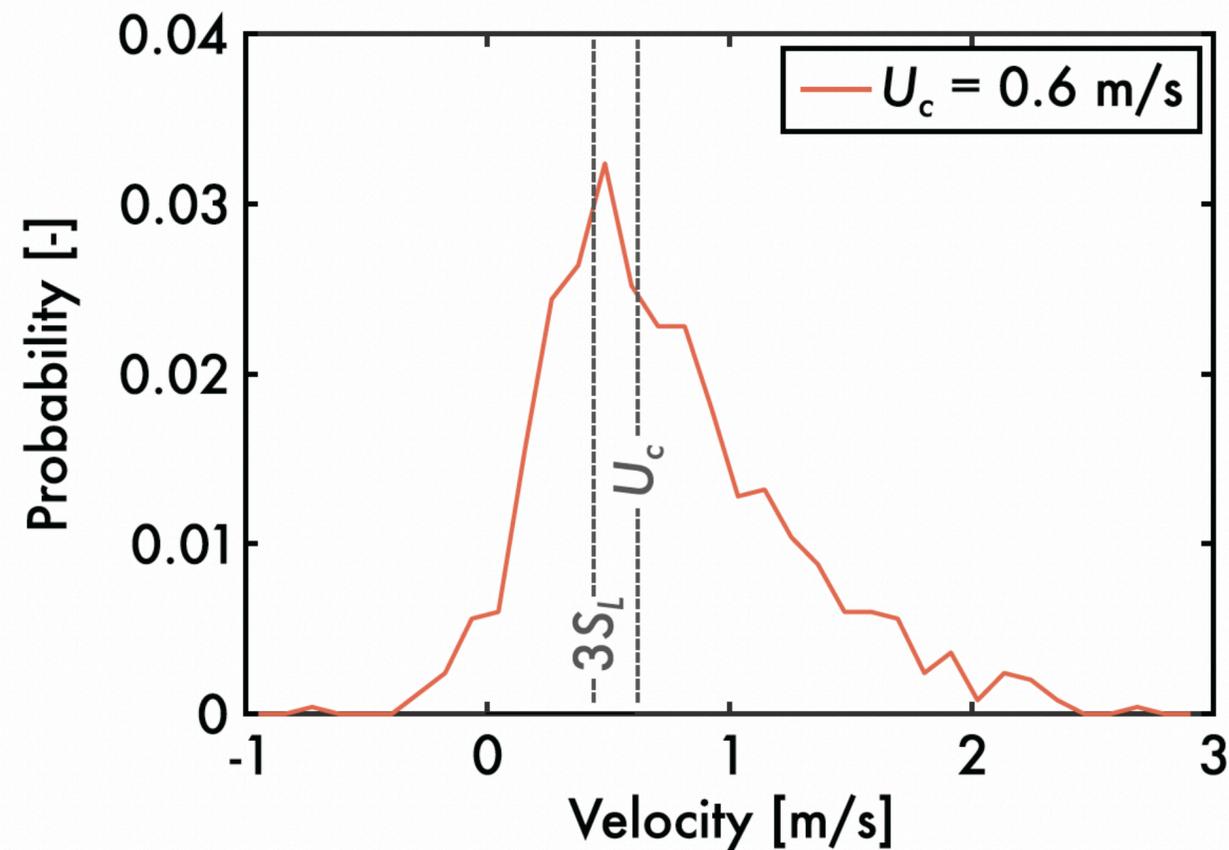
Stabilization mechanisms



If the co-flow velocity is increased to $U_c = 0.6 \text{ m/s}$ ($U_{c,crit} = 0.5 \text{ m/s}$):



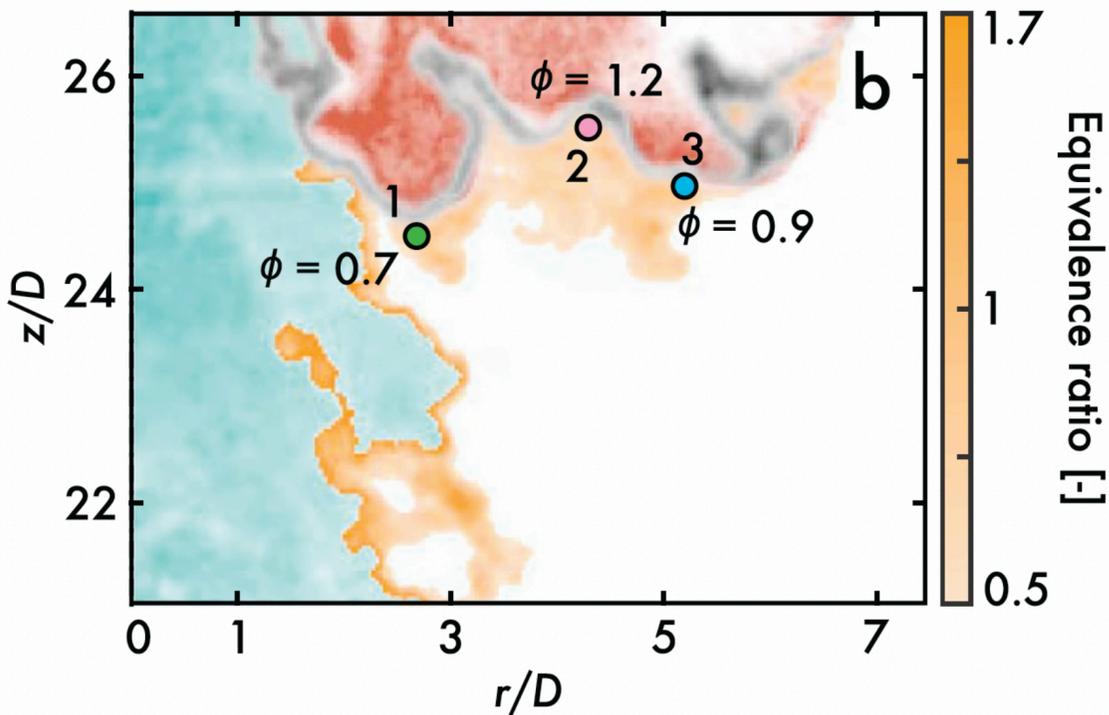
The normal velocity immediately upstream of the flame front may exceed that possible with an edge flame



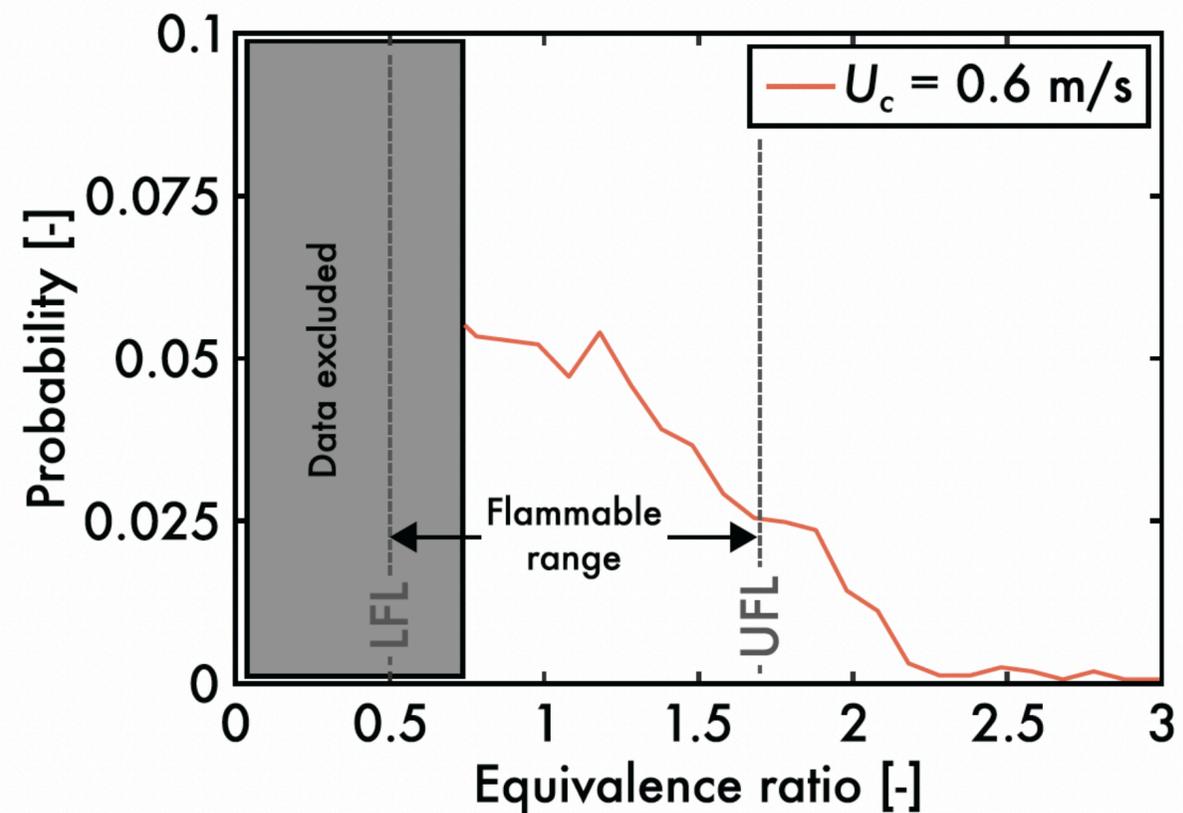
Stabilization mechanisms



If the co-flow velocity is increased to $U_c = 0.6 \text{ m/s}$ ($U_{c,crit} = 0.6 \text{ m/s}$):



The equivalence ratio immediately upstream of the flame front spans the whole flammable range



These observations are consistent with turbulent premixed flame stabilization



Stabilization mechanisms



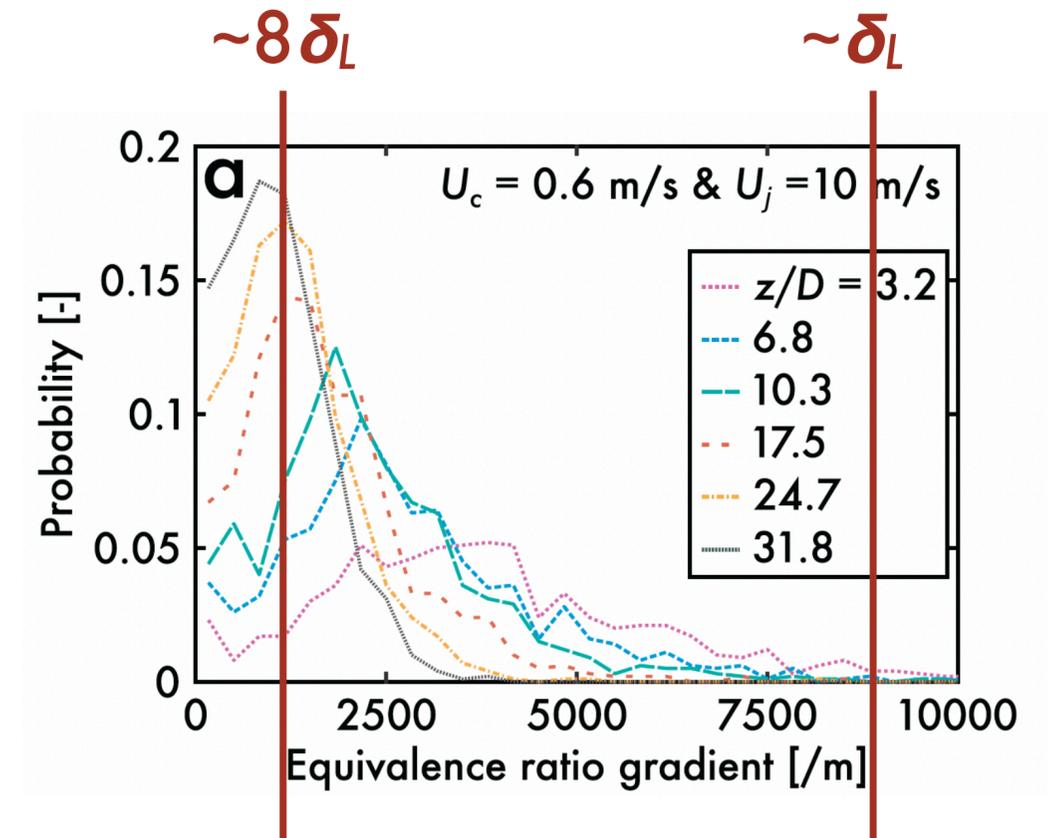
How far from the nozzle does a flame needs to be to exist as a premixed flame?

This can be partially answered by considering gradients of equivalence ratio and the laminar thermal flame thickness δ_L

Turbulent mixing and equivalence ratio gradients provide a lower bound of the lift-off height

The exact lift-off height also depends on the turbulent burning velocity

Spans the flammable range over:



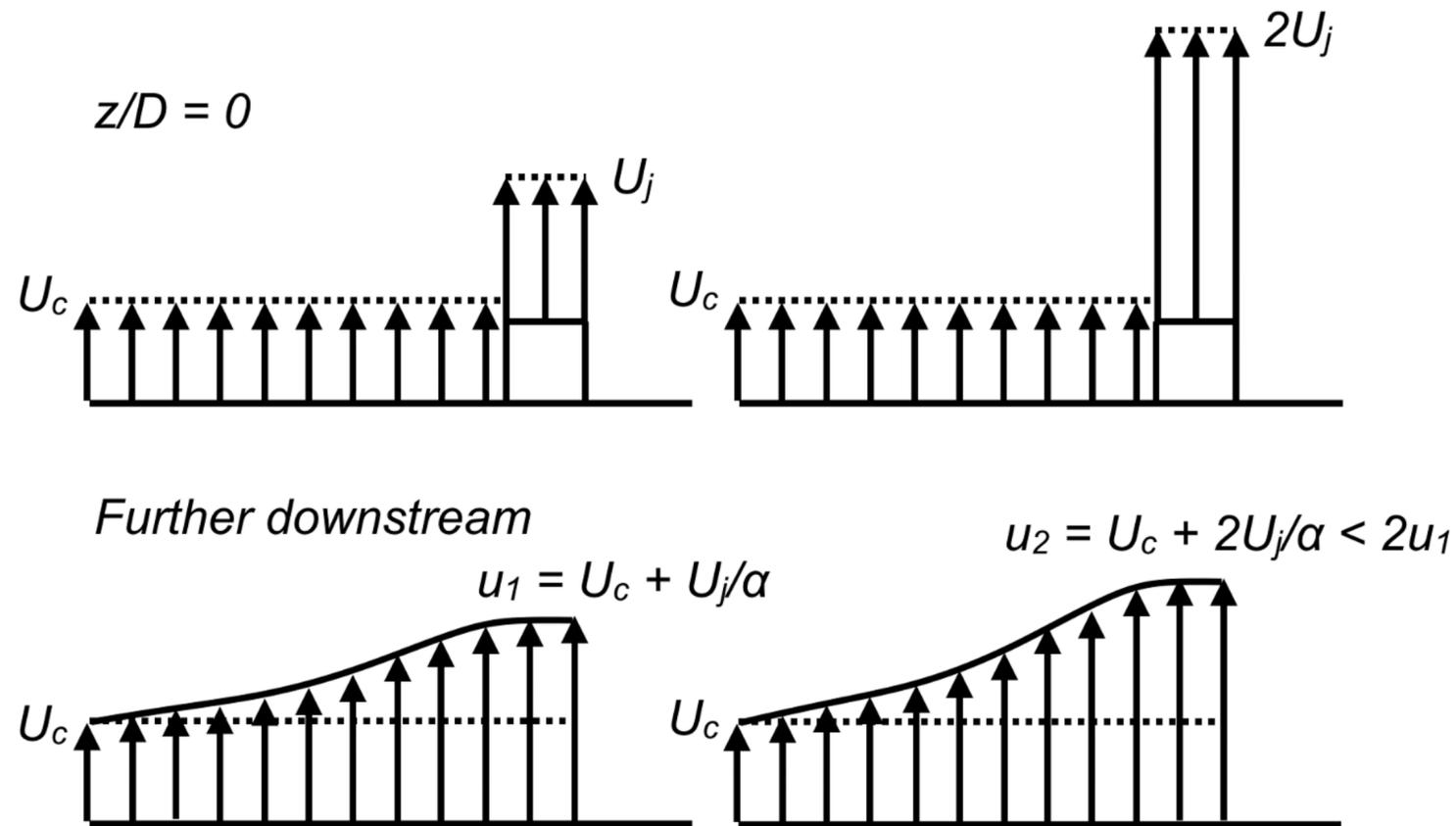
Stabilization mechanisms



The turbulent burning velocity may be calculated using a model such as:

$$S_T = S_L + 0.62S_L^{1/2}u'^{1/2} \left(\frac{u'L_T}{\nu} \right)^{1/4} \propto U_j^{3/4}$$

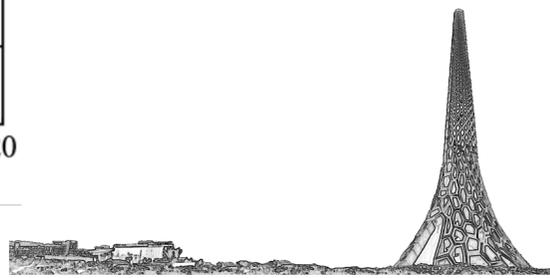
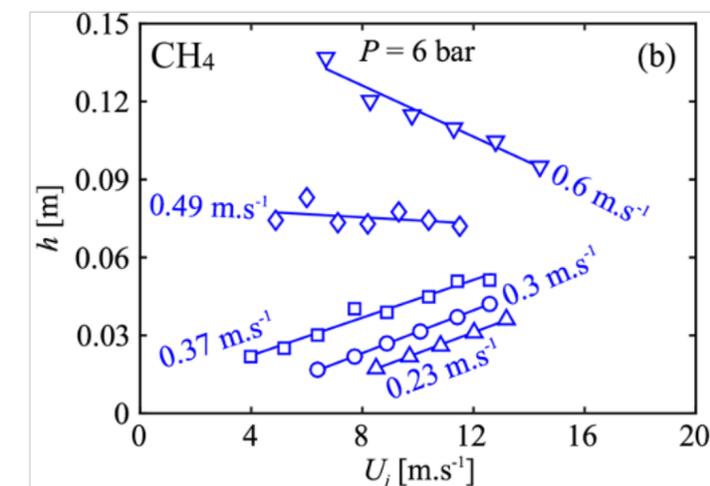
Gulder Proc. Combust. Inst. 1991



Adapted from: Brown et al. Combust. Sci. Technol. 2010

If U_c is sufficiently large, S_T increases faster than the local velocity u when the U_j is increased

➔ The flame propagates closer to the nozzle

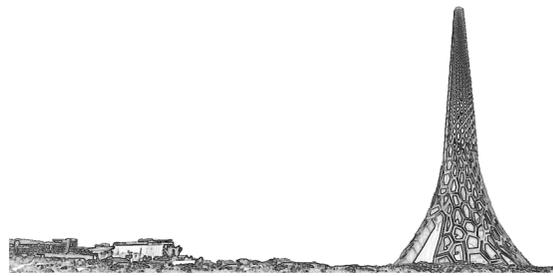




Stabilization mechanisms

Specifically, the effects of pressure are:

- For hydrocarbons, S_L decreases with P . Therefore, the critical co-flow velocity decreases with P too
 - ➔ What about H_2 that features a non-monotonic pressure sensitivity?
- Once pushed downstream, the flame may blow-out, or stabilize as a premixed flame
 - ➔ Increasing P enhances mixing, which helps to stabilize the flame and retards blow off
 - ➔ This is why slope inversion is easier to observe at elevated pressure or with fast fuels (H_2 ?)



Conclusions for the stabilization of non-premixed flames

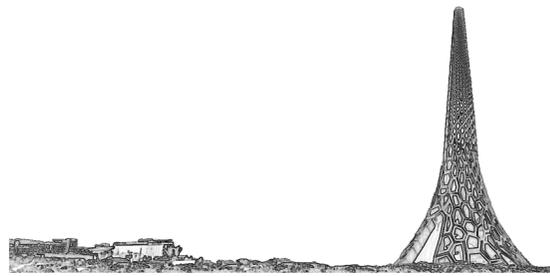


The pressure knob allowed us to refine our understanding of the mechanisms controlling the detachment and lift-off height of turbulent non-premixed flames

We can predict the conditions leading to the detachment of these flames **H₂? Swirl?**

We can predict if lifted flames are propagating as edge flames or premixed flames **H₂? Swirl?**

If they are edge flames, we can predict their lift-off height **H₂? Swirl?**



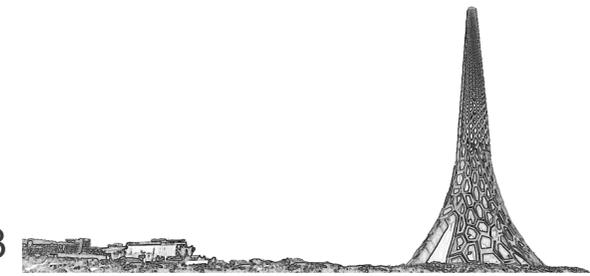
Diagnostics opportunities for H₂ at high pressure



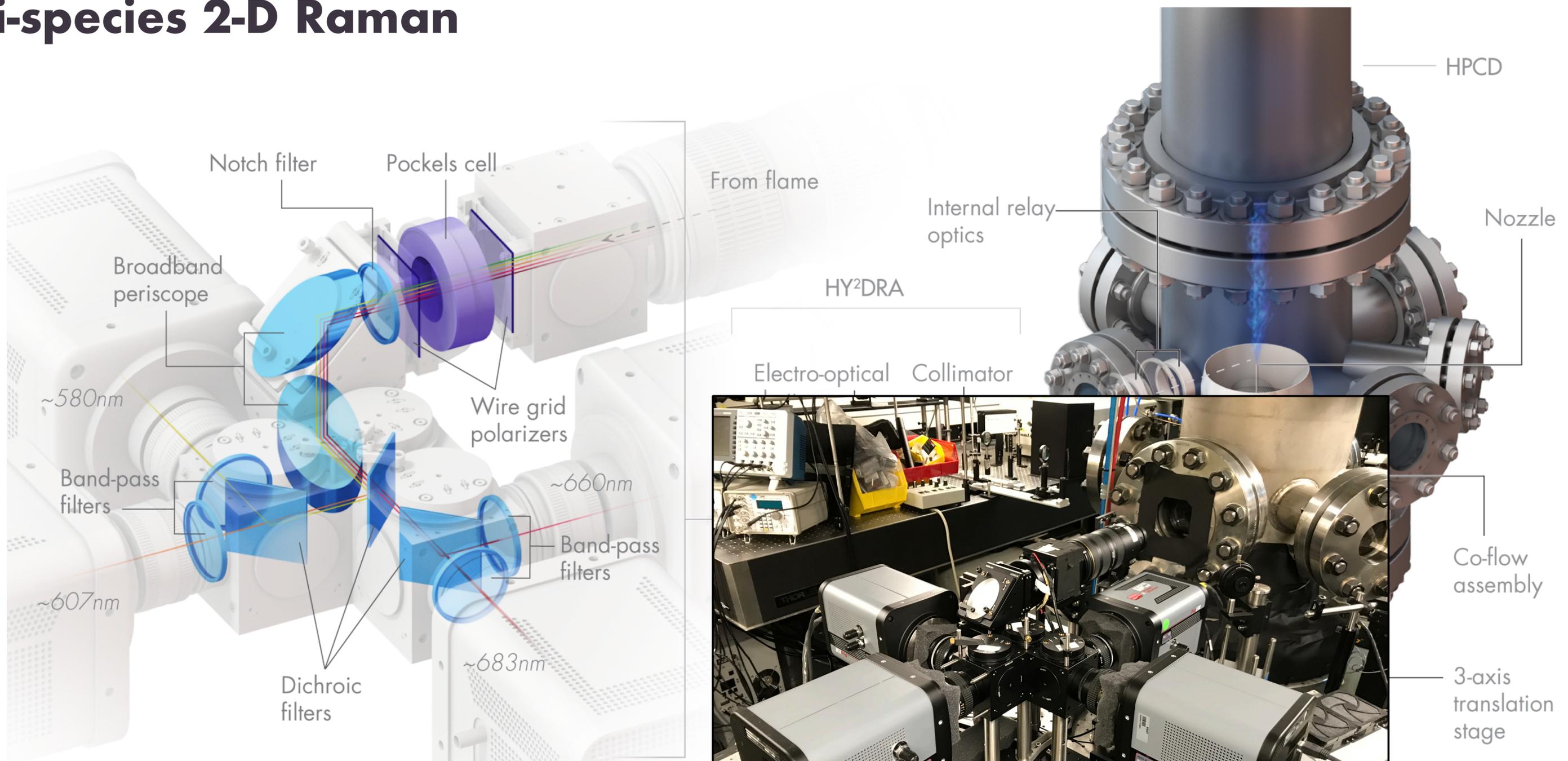
H₂ is a simple fuel: less scalars need to be measured to capture the flame chemistry

The increase of number density due to an increase of pressure may yield more intense signals

- ➔ **Demonstrate the first simultaneous single-shot imaging of all major species, OH, mixture fraction, and temperature in a turbulent flame**

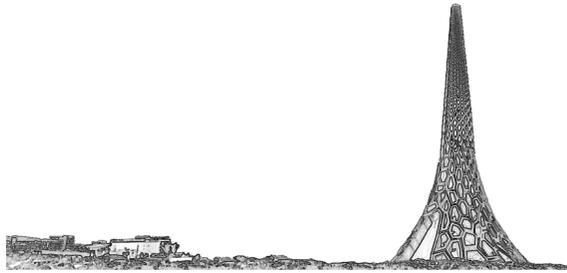


Multi-species 2-D Raman



HY²DRA, a custom signal collection system

- ▶ 4 CCD cameras (N₂, O₂, H₂, H₂O)
- ▶ 4" internal achromatic lenses
- ▶ Pockels cell electro-optical shutter
- ▶ Advanced dichroic mirror assembly



Multi-species 2-D Raman

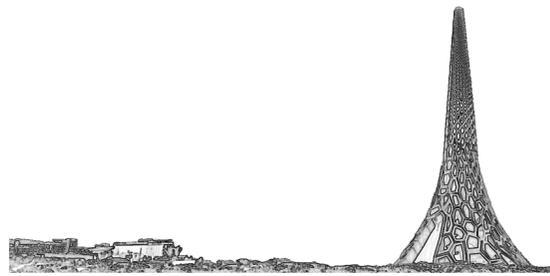


The Raman signal intensity is a function of number density $n(X_k, T)$

Therefore, one has 2 options:

- (1) also **measure temperature** (e.g. with Rayleigh) and derive X_k from single species Raman using the **ideal gas law**
- (2) **measure all major species** with Raman and use additional equation $\sum X_k = 1$ to infer temperature

Because all major species and temperature are measured, **OH-PLIF is made quantitative** by computing **quenching rates** and **Boltzmann population distributions**

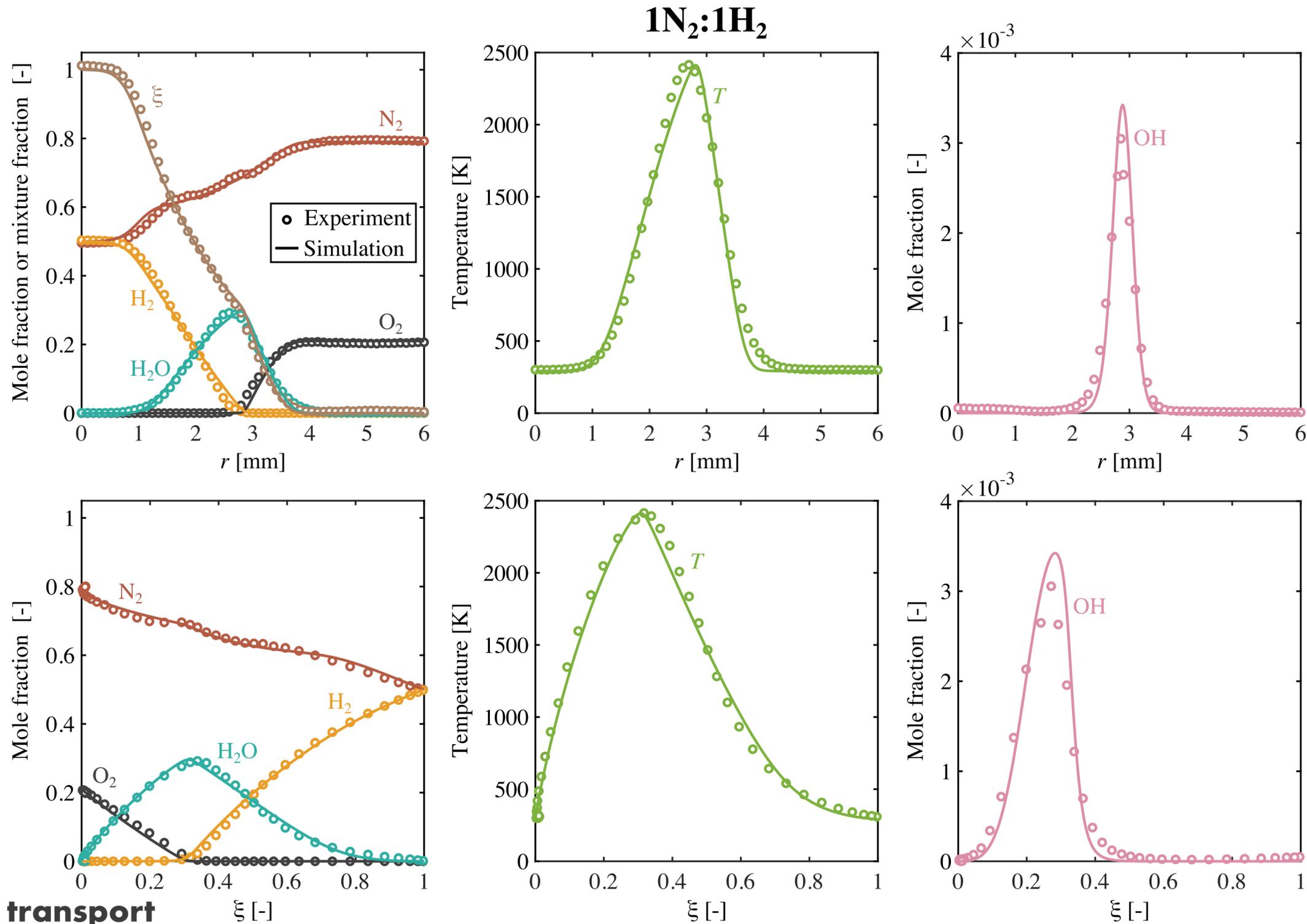


Multi-species 2-D Raman

Results of the calibration in laminar $N_2:H_2$ non-premixed jet flames at 12 bar



1 row of pixels
Averaged over
100 shots



Simulations

- ▶ Chemkin
- ▶ Opposed-flow
- ▶ USCII mech
- ▶ **Multicomponent transport**

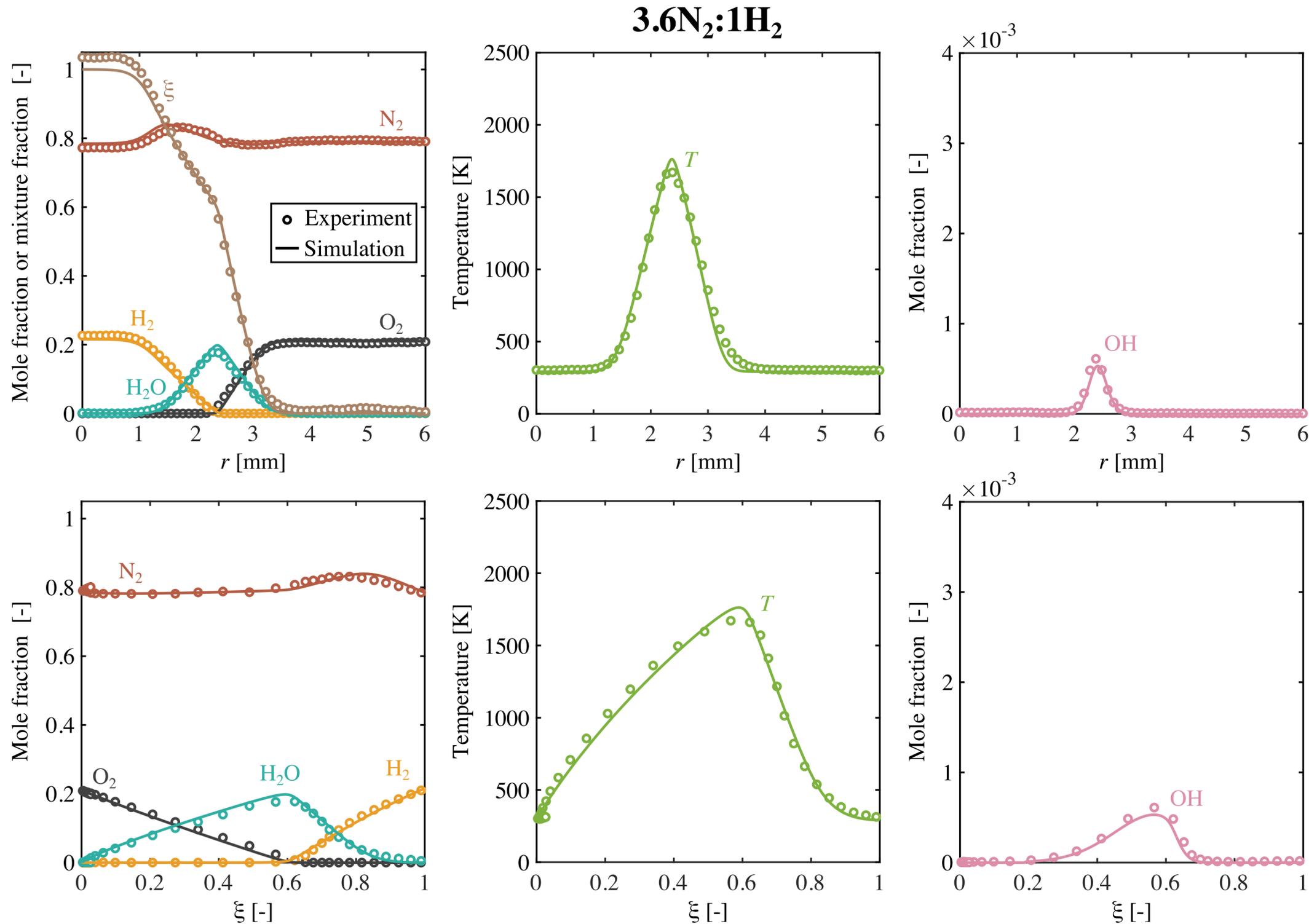


Multi-species 2-D Raman

Results of the calibration in laminar $N_2:H_2$ non-premixed jet flames at 12 bar



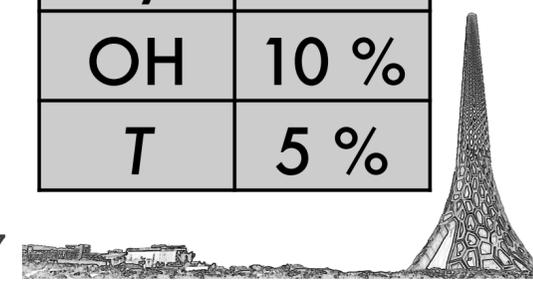
Accuracy



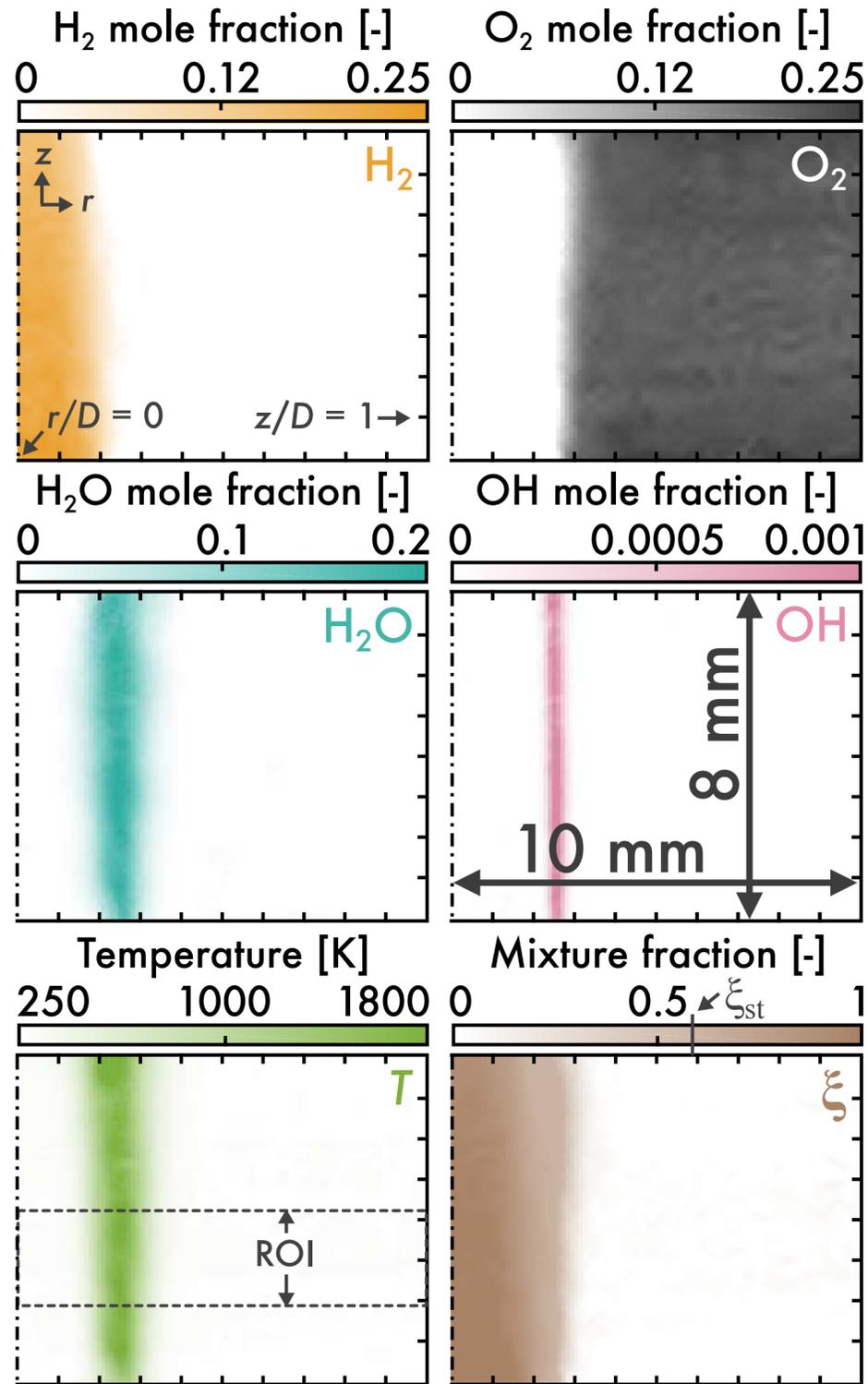
N ₂	2 %
O ₂	2 %
H ₂	4 %
H ₂ O	10 %
ξ	5 %
OH	10 %
T	5 %

Precision

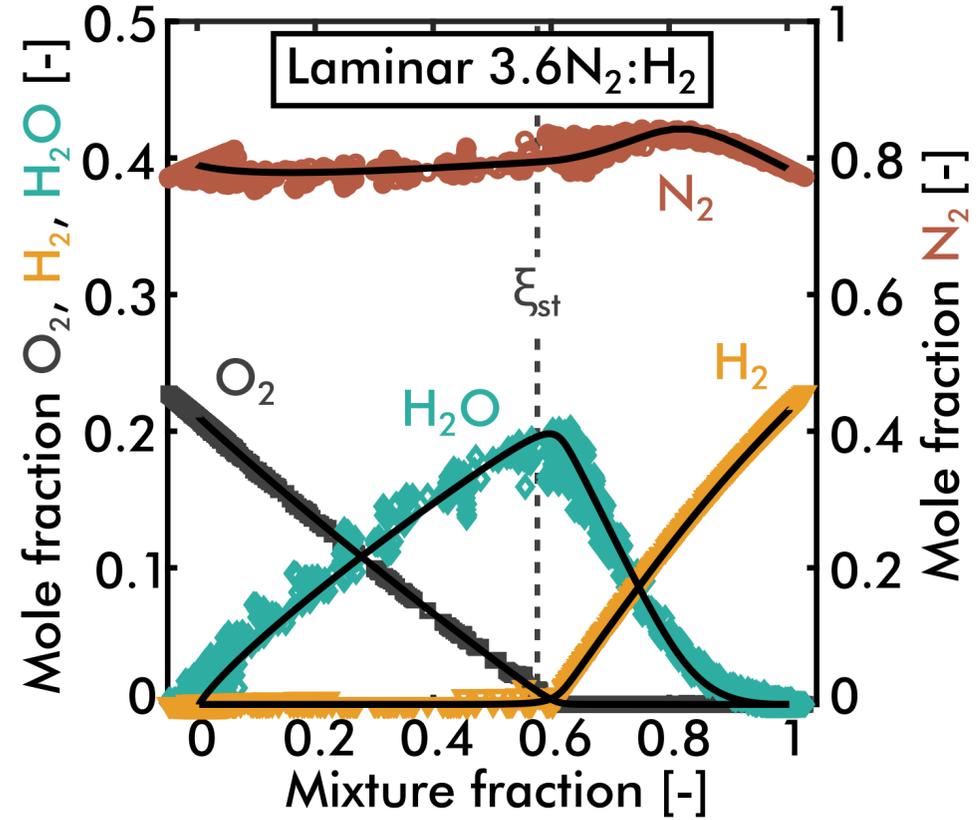
N ₂	2 %
O ₂	3 %
H ₂	4 %
H ₂ O	6 %
ξ	5 %
OH	10 %
T	5 %



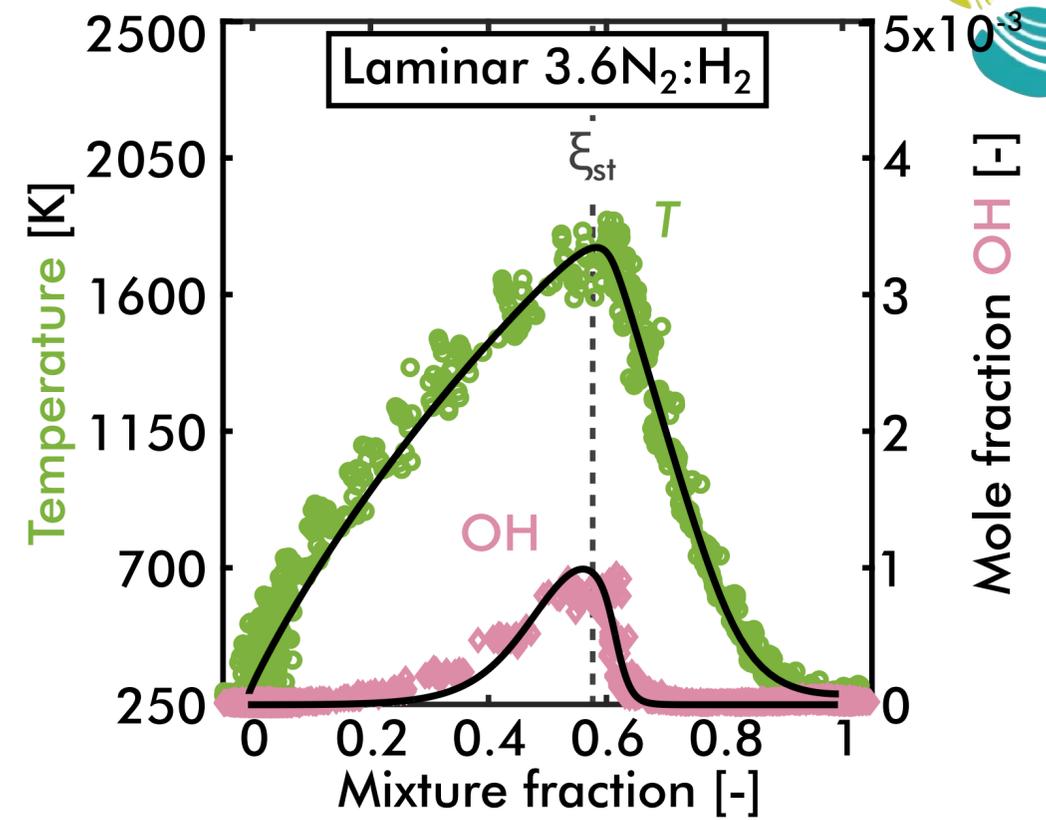
Multi-species 2-D Raman



3.6N₂:1H₂ laminar (single shot)



Single shot - all pixels in ROI



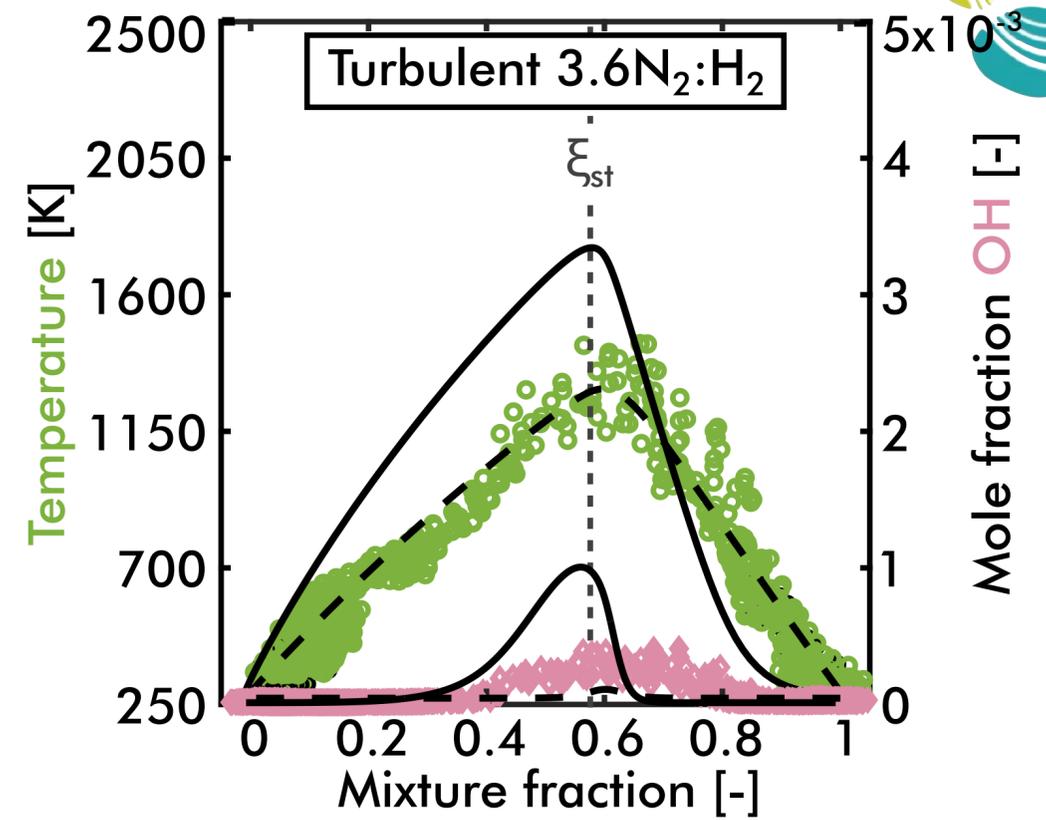
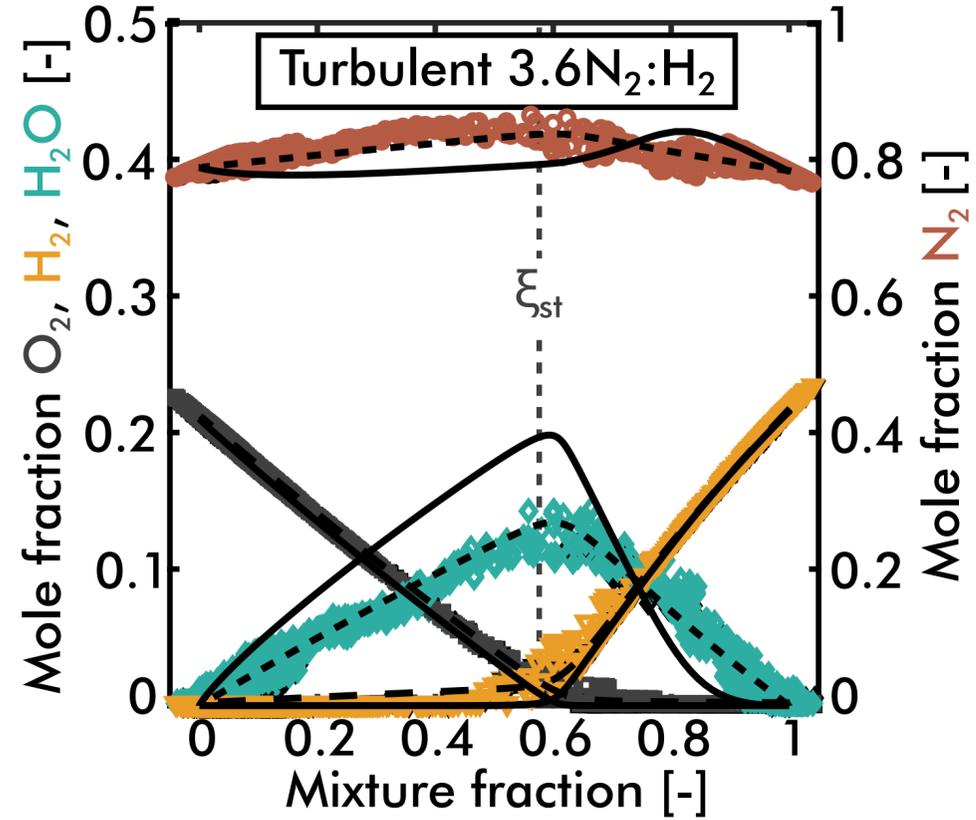
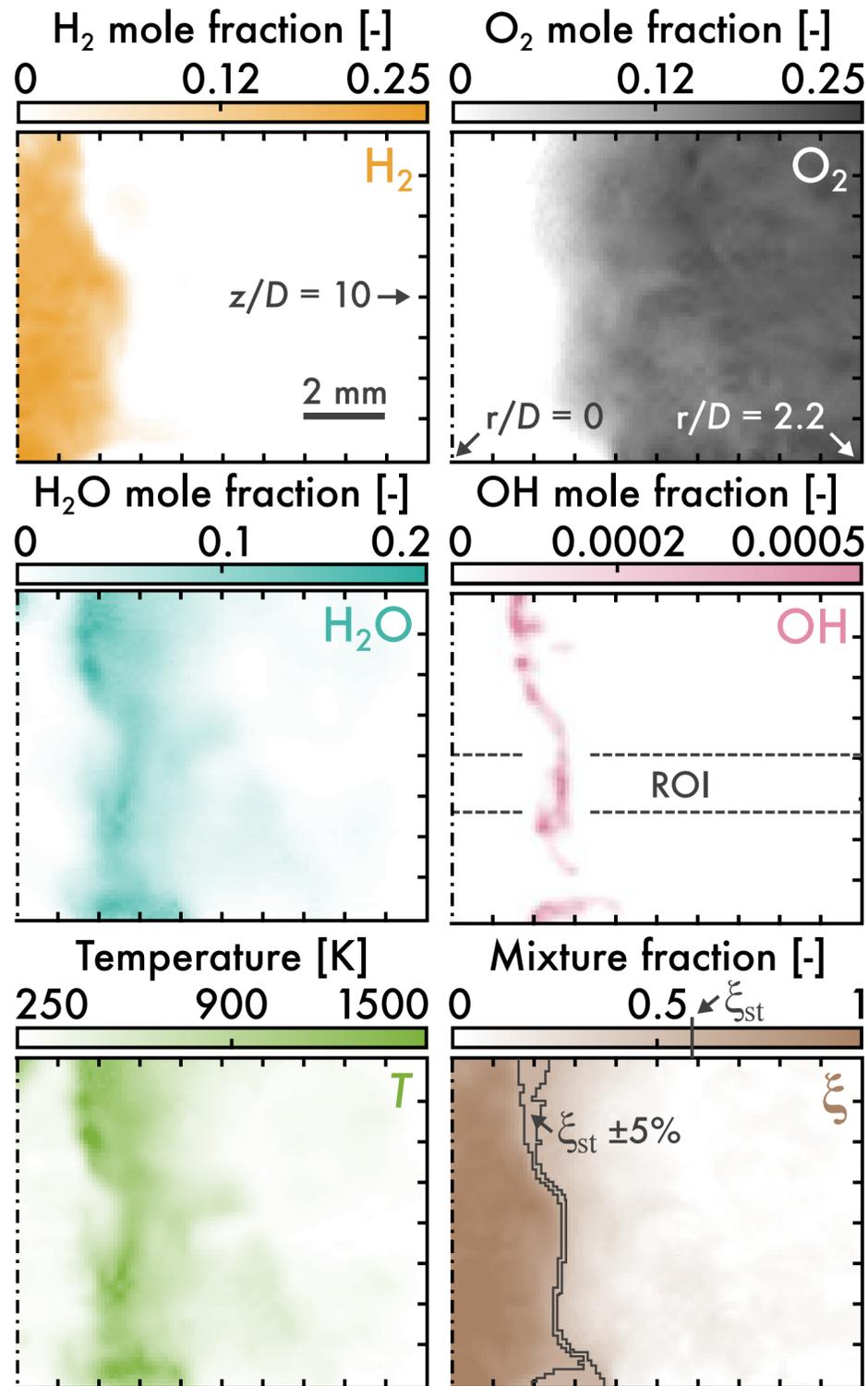
We accurately capture the **full thermo-chemical structure** of the flame in a **single shot**



Mole fraction OH [-]



Multi-species 2-D Raman



Multicomponent transport simulations (—) do not capture experiment in turbulent flames but **Le = 1** simulations (- -) do

- Turbulence smears differential diffusion

This lends confidence in this imaging technique

3.6N₂:1H₂ turbulent (Re = 29,000) (single shot)



Multi-species 2-D Raman



Why is 2-D desired?

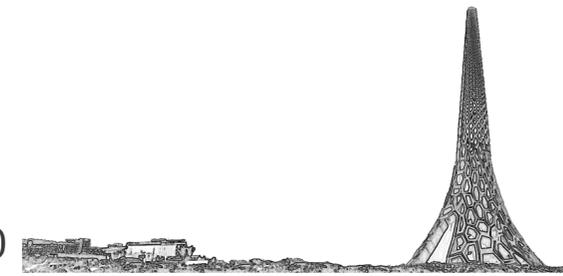
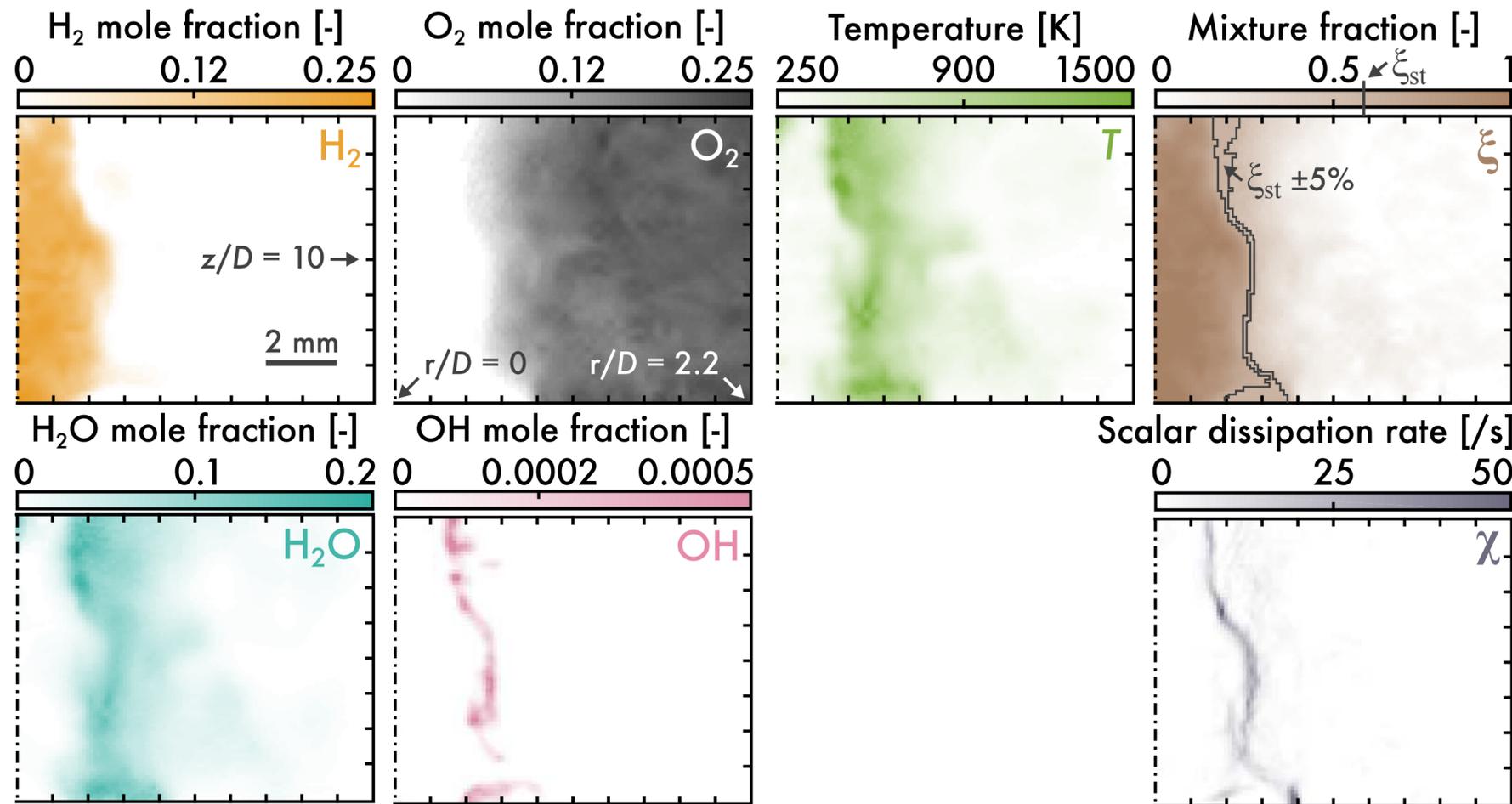
Scalar dissipation rate

$$\chi = 2D (\nabla \xi)^2$$

Speed/cost

Compared to 0-D/1-D, data are collected more rapidly in 2-D

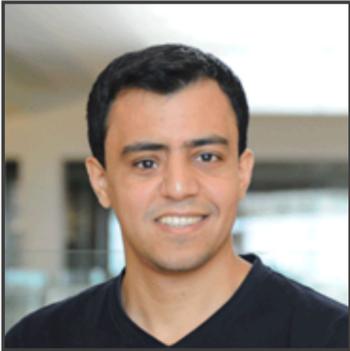
This could be very important for "expensive" or "short" experiments



The high-pressure team



Cristian Avila



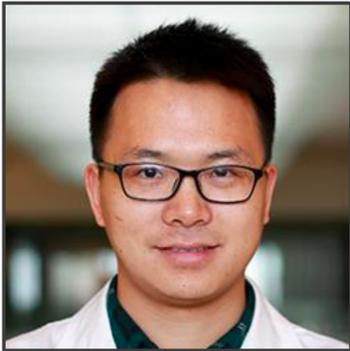
Abdulrahman Alkhateeb



Hao Tang



Francesco Di Sabatino



Xuren Zhu



Guoqing Wang



Yedhu Krishna



Wesley Boyette



Chaobo Yang



Deanna Lacoste



William Roberts



Gaetano Magnotti



Thibault Guiberti

The University of Sydney



Assaad Masri

IMFT



Sylvain Marragou



Laurent Selle



Thierry Poinot



Thierry Schuller





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Science and Technology

H₂ week in Toulouse, February 26th-29th 2024

Questions?

